



## Calhoun: The NPS Institutional Archive DSpace Repository

---

Theses and Dissertations

1. Thesis and Dissertation Collection, all items

---

1973

### Digital computer simulation for surface ship control.

Ratanaruang, Aporn.

Monterey, California. Naval Postgraduate School

---

<http://hdl.handle.net/10945/16567>

---

*Downloaded from NPS Archive: Calhoun*



<http://www.nps.edu/library>

Calhoun is the Naval Postgraduate School's public access digital repository for research materials and institutional publications created by the NPS community.

Calhoun is named for Professor of Mathematics Guy K. Calhoun, NPS's first appointed -- and published -- scholarly author.

Dudley Knox Library / Naval Postgraduate School  
411 Dyer Road / 1 University Circle  
Monterey, California USA 93943

DIGITAL COMPUTER SIMULATION  
FOR SURFACE SHIP CONTROL

Aporn Ratanaruang

Library  
Naval Postgraduate School  
Monterey, California 93940

# NAVAL POSTGRADUATE SCHOOL

## Monterey, California



# THESIS

DIGITAL COMPUTER SIMULATION  
FOR  
SURFACE SHIP CONTROL

by

Aporn Ratanaruang

Thesis Advisor:

M. L. Wilcox

June 1973

Approved for public release; distribution unlimited.

T155110



Digital Computer Simulation  
for  
Surface Ship Control

by

Aporn Ratanaruang  
Lieutenant Commander, Royal Thai Navy  
Thai Naval Academy, 1957  
B.S.E.E., Naval Postgraduage School, 1973

Submitted in partial fulfillment of the  
requirements for the degree of

MASTER OF SCIENCE IN ELECTRICAL ENGINEERING

from the  
NAVAL POSTGRADUATE SCHOOL  
June 1973



## ABSTRACT

The general equations of surface ship motion are developed and standardized for simulation in digital computer. Digital simulations of the dynamics of the surface ship in three degrees of freedom are done with and without non-linear and cross-coupling terms.



## TABLE OF CONTENTS

I.	EQUATIONS OF SURFACE SHIP MOTION -----	6
II.	DIGITAL COMPUTER SIMULATION -----	15
III.	STUDY OF SHIP "D" PERFORMANCE -----	27
	A. NON-LINEAR TERMS NOT INCLUDED -----	30
	B. INCLUDED NON-LINEAR TERMS -----	44
IV.	CONCLUSIONS -----	63
	COMPUTER PROGRAM -----	32
1.	SURFACE SHIP 3 DEGREES OF FREEDOM WITHOUT NON-LINEAR TERMS -----	32
2.	SURFACE SHIP 3 DEGREES OF FREEDOM WITH NON-LINEAR TERMS -----	46
3.	SUBROUTINE TO CALL DRAW -----	61
	LIST OF REFERENCES -----	64
	INITIAL DISTRIBUTION LIST -----	65
	FORM DD 1473 -----	66



## LIST OF TABLES

I.	Hydrodynamic coefficients and constants of ship "D" for linear terms -----	30
II.	Hydrodynamic coefficients of ship "D" for non-linear terms -----	45



#### ACKNOWLEDGEMENT

The author wishes to express his sincere appreciation to Professor MILTON L. WILCOX for the guidance, assistance and continuous encouragement which he provided during the pursuit of this study. The author would also like to express his appreciation to Dr. GEORGE J. THALER for his valuable assistance in furnishing the data for simulation.



## I. EQUATIONS OF SURFACE SHIP MOTION

A moving ship is a body with six degrees of freedom. These degrees of freedom are generally chosen as follows:

a. Linear displacements along the three axes through the center of gravity.

- a.1 Surge -- along X axes
- a.2 Sway -- along Y axes
- a.3 Heave -- along Z axes

b. Rotations around the three axes through the center of gravity.

- b.1 Roll -- around X axes
- b.2 Pitch -- around Y axes
- b.3 Yaw -- around Z axes

Further reduction in the complex nature of the equations can be brought about by choosing an orthogonal axis system parallel to the principal axes of inertia so as to eliminate products of inertia in the motion equations. For practically all ocean vehicles, with extremely few exceptions, a longitudinal axis (X axis) in the centerline plane, a downward (toward keel) axis (Z axis) perpendicular to the X axis in the centerline plane, and a transverse axis (Y axis) perpendicular to the centerline plane satisfies this requirement.



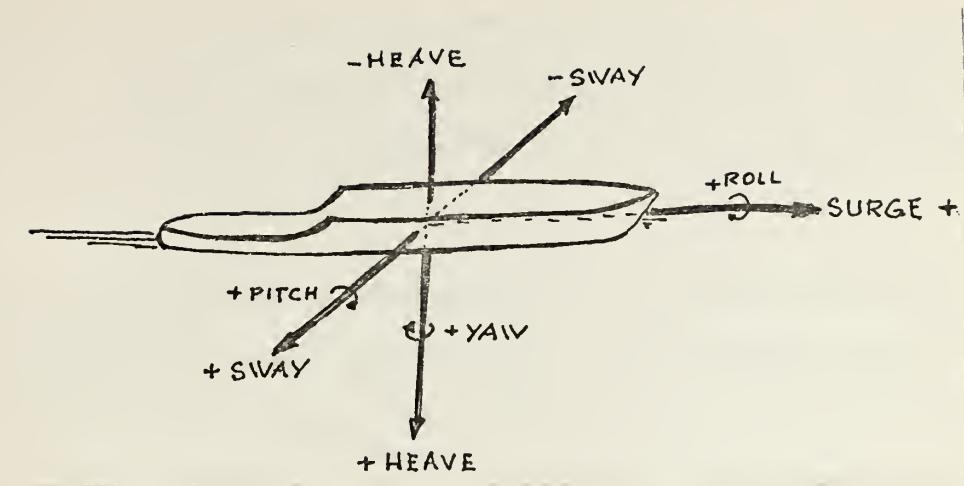


Figure 1. Surface ship in six degrees of freedom.

For the exceptional vehicle which has a very peculiar and significantly large asymmetrical mass distribution, it is necessary to include the products of inertia.

The X, Y, Z axes form an orthogonal right hand system of axes fixed in the vehicle. The axes and the associated components of the pertinent physical quantities are defined below:

The longitudinal X axis (in the plane of symmetry) is positive in the forward direction, usually parallel to the keel or calm water line. If the upper and lower halves of the body are symmetrical, then the axis is the intersection of the two planes of symmetry.

The Y axis is the transverse axis perpendicular to the plane of symmetry and positive to the starboard.

The Z axis or downward axis in the plane of symmetry (X, Z) is perpendicular to the X axis and positive downward towards the keel.



$\hat{i}, \hat{j}, \hat{k}$  unit vectors along the X, Y, Z axis respectively.

$\vec{R}$  x, y, z vector distance of a point from the origin O, and the corresponding components along the X, Y and Z axes.

$$\vec{R} = i x_G + j y_G + k z_G$$

$\vec{U}$  u, v, w velocity of the origin O (on the body) and the corresponding components along the X, Y and Z axis.

$$\vec{U} = \hat{i} u + \hat{j} v + \hat{k} w$$

$\vec{\Omega}$  p, q, r angular velocity of the body about the origin and the corresponding components about the axes.

$$\vec{\Omega} = \hat{i} p + \hat{j} q + \hat{k} r$$

The moments of inertia of the body about the X, Y and Z axes respectively  $I_x, I_y, I_z$ .

$\vec{F}$  X, Y, Z, force acting on the body and the corresponding components along the axes.

$$\vec{F} = \hat{i} F_x + \hat{j} F_y + \hat{k} F_z$$

$\vec{m}$  K, M, N Moments acting about the axes.

$$\vec{m} = \hat{i} K + \hat{j} M + \hat{k} N$$

Newton's law of motion for a rigid body can be written as two equations, one a force equation and the second a moment equation provided an origin is taken at the center of gravity and the axis system is fixed in space. The equations are

$$\vec{F} = \frac{d}{dt} \xrightarrow{\text{momentum}} \cdot \frac{d}{dt} (m \vec{U}_G)$$

$$\vec{m} = \frac{d}{dt} \xrightarrow{\text{angular momentum}}_G = \frac{d}{dt} (I \vec{\Omega})$$



where the subscript G refers to an origin at the center of gravity and m is the mass of the body. For a mass essentially constant in time

$$\vec{F} = m \frac{d}{dt}(\vec{U}_G)$$

For an origin not at the center of gravity of the body and in a system of axes fixed in and moving with the vehicle.

$$\vec{U}_G = U_a + \vec{\Omega} \times \vec{R}_G$$

where  $U_a$  is the velocity of the origin in space. However, since the origin is on the surface of the earth and the earth rotates, then

$$\vec{U}_a = \vec{U} + \vec{\Omega}_e \times \vec{R}_b$$

where  $\vec{U}$  is the geographical velocity of the body,  $\vec{\Omega}_e$  is the angular velocity of the earth, and  $\vec{R}_b$  is the radius vector from earth's center to the vehicle. The force equation becomes:

$$\vec{F} = m \frac{d}{dt}(\vec{U} + \vec{\Omega}_e \times \vec{R}_b + \vec{\Omega} \times \vec{R}_G)$$

$$\vec{F} = m \frac{d}{dt}(\vec{U} + \vec{\Omega} \times \vec{R}_b) + m[\vec{\Omega}_e \times \vec{R}_b + \vec{\Omega}_e \times \vec{\Omega}_e \times \vec{R}_b]$$

$$= m \frac{d}{dt}(\vec{U} + \vec{\Omega} \times \vec{R}_G) + m[\vec{\Omega}_e \times \vec{U} + \vec{\Omega}_e \times \vec{\Omega}_e \times \vec{R}_b]$$

since  $\vec{\Omega} = 0$  and  $\vec{R}_b = \vec{U} + \vec{\Omega}_e \times \vec{R}_b$

the term  $m\vec{\Omega}_e \times \vec{U}$  is the coriolis force and  $m\vec{\Omega}_e \times \vec{\Omega}_e \times \vec{R}_b$  is the centripetal acceleration due to rotation of the



earth. These two terms are negligibly small when compared with the other forces, then

$$\vec{F} = m \frac{d}{dt} (\vec{U} + \vec{\omega} \times \vec{R}_G)$$

Finding the derivatives of unit vectors (change in direction)

$$\begin{aligned} \text{where } \hat{di} &= -\hat{k}d\theta & \hat{di} &= \hat{j}d\psi & \hat{di} &= 0 \\ \hat{dj} &= 0 & \hat{dj} &= -\hat{i}d\psi & \hat{dj} &= \hat{k}d\phi \\ \hat{dk} &= \hat{i}d\theta & \hat{dk} &= 0 & \hat{dk} &= -\hat{j}d\phi \end{aligned}$$

Adding the contributions

$$\begin{aligned} \hat{\frac{di}{dt}} &= \hat{i} \cdot 0 + \hat{j} \frac{d\psi}{dt} - \hat{k} \frac{d\theta}{dt} \\ \hat{\frac{dj}{dt}} &= -\hat{i} \frac{d\psi}{dt} + \hat{j} \cdot 0 + \hat{k} \frac{d\phi}{dt} \\ \hat{\frac{dk}{dt}} &= \hat{i} \frac{d\theta}{dt} - \hat{j} \frac{d\phi}{dt} + \hat{k} \cdot 0 \\ \vec{\omega} &= \hat{i}p + \hat{j}q + \hat{k}r \end{aligned}$$

$$p = \frac{d\phi}{dt}, \quad q = \frac{d\theta}{dt}, \quad r = \frac{d\psi}{dt}$$

$$\hat{\frac{di}{dt}} = \vec{\omega} \times \hat{i} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ p & q & r \\ 1 & 0 & 0 \end{vmatrix}$$

$$\vec{U} = \hat{i}u + \hat{j}v + \hat{k}w$$

$$\vec{R}_G = \hat{i}x_G + \hat{j}y_G + \hat{k}z_G$$

$$\begin{aligned} \hat{\frac{dU}{dt}} &= \hat{i}\dot{u} + u(\hat{j}\frac{d\hat{i}}{dt} + \hat{j}\dot{v} + v\frac{d\hat{j}}{dt} + \hat{k}\dot{w} + w\frac{d\hat{k}}{dt}) \\ &= \hat{i}\dot{u} + u(\hat{j}\frac{d\psi}{dt} - \hat{k}\frac{d\theta}{dt}) + \hat{j}\dot{v} + v(-\hat{i}\frac{d\psi}{dt} + \hat{k}\frac{d\phi}{dt}) \\ &\quad + \hat{k}\dot{w} + w(\hat{i}\frac{d\theta}{dt} - \hat{j}\frac{d\phi}{dt}) \\ &= \hat{i}\dot{u} + u(\hat{j}r - \hat{k}q) + \hat{j}\dot{v} + v(-ir + \hat{k}p) + \hat{k}\dot{w} + w(\hat{i}q - \hat{j}p) \end{aligned}$$



$$\vec{\Omega} \times \vec{R}_G = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ p & q & r \\ x_G & y_G & z_G \end{vmatrix} = \hat{i}(qz_G - ry_G) - \hat{j}(pz_G - rx_G) + \hat{k}(py_G - qx_G)$$

$$\begin{aligned} \frac{d}{dt}(\vec{\Omega} \times \vec{R}_G) &= \hat{i}(z_G \dot{q} - y_G \dot{r}) + \hat{j}(z_G qr - y_G r^2) + \hat{k}(y_G qr - z_G q^2) \\ &+ \hat{j}(x_G \dot{r} - z_G \dot{p}) + \hat{k}(x_G rp - z_G p^2) + \hat{i}(z_G rp - x_G r^2) \\ &+ \hat{k}(y_G \dot{p} - x_G \dot{q}) + \hat{i}(y_G pq - x_G q^2) + \hat{j}(x_G pq - y_G p^2) \end{aligned}$$

from  $\vec{F} = \hat{i}x + \hat{j}y + \hat{k}z$  and  $\vec{F} = m[\frac{d\vec{U}}{dt} + \frac{d}{dt}(\vec{\Omega} \times \vec{R}_G)]$

Rewriting and grouping all  $\hat{i}$  terms equal to  $X$ , all  $\hat{j}$  terms equal to  $Y$  and all  $\hat{k}$  terms equal to  $Z$  yield

$$\begin{aligned} X &= m(\dot{u} - vr + wg + z_G \dot{q} + rpz_G - r^2 x_G + pqy_G - q^2 x_G) \\ &= m(\dot{u} + wg - vr - x_G(r^2 + q^2) + y_G(pq - \dot{r}) + z_G(rp + \dot{q})) \\ Y &= m(\dot{v} + ur - wp + x_G(pq + \dot{r}) - y_G(p^2 + r^2) + z_G(pq - \dot{r})) \quad (1) \\ Z &= m(\dot{w} + vp - uq + x_G(rp - \dot{q}) + y_G(qr + \dot{p}) - z_G(p^2 + q^2)) \end{aligned}$$

From  $\vec{m}_G = \frac{d}{dt}(\text{angular momentum})_G$

$$\begin{aligned} \vec{m}_G &= \frac{d}{dt}(\hat{i}I_{x_G}p + \hat{j}I_{y_G}q + \hat{k}I_{z_G}r) \\ &= \frac{d}{dt}(\hat{i}I_{x_G}p + \hat{j}I_{y_G}q + \hat{k}I_{z_G}r) \end{aligned}$$

$G$  indicates an origin at the center of gravity

$$\begin{aligned} I_{x_G} &= I_x - m(y_G^2 + z_G^2) \\ I_{y_G} &= I_y - m(z_G^2 + x_G^2) \\ I_{z_G} &= I_z - m(x_G^2 + y_G^2) \end{aligned}$$

$$\vec{M} = \vec{m}_G + \vec{R}_G \times \vec{F} \quad \text{or} \quad \vec{M}_G = \vec{M} - \vec{R}_G \times \vec{F}$$



After manipulating and using the results for the derivatives of unit vectors (same as above) expressions for K, H and N are obtained.

$$\begin{aligned} K &= I_X \dot{p} + (I_Z - I_Y) qr + m [Y_G (w + pv - qu) - Z_G (\dot{v} + ru - pw)] \\ M &= I_Y \dot{q} + (I_X - I_Z) rp + m [Z_G (u + qw - rv) - X_G (w + pv - qu)] \\ N &= I_Z \dot{r} + (I_Y - I_X) pq + m [X_G (v + ru - pw) - Y_G (u + qw - rv)] \end{aligned} \quad (2)$$

The terms  $(qw - rv)$ ,  $(ru - pw)$  and  $(pv - qu)$  are gyroscopic effects.

The relationship for forces and moments can be expressed

$$\begin{aligned} \text{Force} \\ \text{Moment} \end{aligned} \quad ) = f(\text{properties of body, properties of motion, properties of fluid})$$

For a particular ship, in a given fluid with no excitation force - so

$$\begin{aligned} \vec{F} \\ \vec{M} \end{aligned} \quad ) = f(\text{properties of motion}) \\ = f(X_O, Y_O, Z_O, \phi, \theta, \psi, u, v, w, p, q, r, \dot{u}, \dot{v}, \dot{w}, \dot{p}, \dot{q}, \dot{r}, \delta, \delta', \delta'')$$

The Taylor series which has the following form may now be applied to linearize the equations about an operating point  $\bar{X}_O$

$$f(x) = f(\bar{X}_O) + (x - \bar{X}_O) \frac{d}{dx} f(\bar{X}_O) + (x - \bar{X}_O)^2 \frac{d^2}{dx^2} f(\bar{X}_O) + \dots$$

Apply this to  $f(x, y, z) \dots$



For the case of  $f$  (properties of motion) let  $(X - \bar{X}_o)$   
 $= \Delta X_o$ ,  $(Y - \bar{Y}_o) = \Delta Y_o$  and  $(Z - Z_o) = \Delta Z_o$

terms second order and higher are neglected for small perturbations.

From  $X$  equation the linear terms are obtained:

$$X = f(\dots)_o + \Delta X_o \left( \frac{\delta f}{\delta X_o} \right) + \Delta Y_o \left( \frac{\delta f}{\delta Y_o} \right) + \Delta Z_o \left( \frac{\delta f}{\delta Z_o} \right) + \dots \Delta v \left( \frac{\delta f}{\delta v} \right) + \dots$$

The defining relations are:

$$\left( \frac{\delta f}{\delta u} \right)_o = \left( \frac{\delta X}{\delta u} \right)_{u=u_o} = X_u$$

$$\left( \frac{\delta X}{\delta w} \right)_{w=w_o} = X_w$$

$$\Delta w = (w - w_o) = w, \quad w_o = 0$$

$$\Delta u = (u - u_o)$$

The force equations then become

$$\begin{aligned} X &= X_o + X_{X_o} X_o + X_{Y_o} Y_o + \dots X_{\theta} \theta + \dots X_u \Delta u \\ Y &= Y_o + Y_{X_o} X_o + Y_{Y_o} Y_o + \dots Y_{\theta} \theta + \dots Y_u \Delta u \\ Z &= Z_o + Z_{X_o} X_o + Z_{Y_o} Y_o + \dots Z_{\theta} \theta + \dots Z_u \Delta u \end{aligned} \quad (3)$$

A similar derivation can be done for the  $K, M$  and  $N$  equations. The preceding  $X, Y$  and  $Z$  equations may now be equated to the linearized  $X, Y$  and  $Z$  (equations (1)), e.g., for the  $Y$  equation without roll, pitch, and the center of gravity at  $X_G = 0$ ,  $Y_G = 0$  and  $Z_G = 0$  gives the linearized equation.

$$Y = m(v + ur)$$



then

$$Y_{X_o} \dot{X}_o + Y_{Y_o} \dot{Y}_o + Y_\psi \dot{\psi} + Y_u \dot{u} + Y_v \dot{v} + Y_r \dot{r} + Y_{\dot{u}} \dot{u} + Y_{\dot{v}} \dot{v} + Y_{\dot{r}} \dot{r} = m(\dot{v} + ru)$$

Expressions for  $X, Z$  and  $K, M$  and  $N$  can be determined in a similar procedure.

In order to obtain the equations in a non-dimensional form some definitions will be given, and applied to the  $Y$  force equations as an example of the process.

$$\text{Froude number} = \frac{U}{\sqrt{g\ell}}$$

$$u', v', w' = \frac{u, v, w}{\sqrt{g/\ell}}$$

$$t' = t(\sqrt{g/\ell})$$

$$x', y', z' = \frac{x, y, z}{\rho g \ell^3}$$

$$c_x', c_y', c_z' = \frac{x, y, z}{\frac{1}{2} \rho U^2 \ell^2}$$

After replacing and adding the effect of waves, the  $Y$  equation becomes:

$$\begin{aligned} \dot{v}' + r'u' &= \frac{1}{2} U^2 (c_{Y_v} v' + c_{Y_p} p' + c_{Y_r} r' + c_{Y_{\delta r}} \delta r) \\ &+ (Y_{p'} \dot{p}' + Y_{v'} \dot{v}' + Y_{r'} \dot{r}' + Y_w') \end{aligned}$$



### III. DIGITAL COMPUTER SIMULATION

The six equations of motion after rearranging by placing the second order terms to the left and the rest on the right side become:

$$\begin{aligned} \ddot{aaA} + \ddot{baB} + \ddot{caC} + \ddot{daD} + \ddot{eaE} + \ddot{faF} &= - (a_1 \dot{a}_1 \dot{A} + a_2 \dot{a}_2 \dot{A} + b_1 \dot{a}_1 \dot{B} + b_2 \dot{a}_2 \dot{B} \\ &\quad + c_1 \dot{a}_1 \dot{C} + c_2 \dot{a}_2 \dot{C} + d_1 \dot{a}_1 \dot{D} + d_2 \dot{a}_2 \dot{D} \\ &\quad + e_1 \dot{a}_1 \dot{E} + e_2 \dot{a}_2 \dot{E} + f_1 \dot{a}_1 \dot{F} + f_2 \dot{a}_2 \dot{F}) \\ &\quad + \text{IF1} \end{aligned}$$

$$\begin{aligned} \ddot{abA} + \ddot{bbB} + \ddot{cbC} + \ddot{dbD} + \ddot{ebE} + \ddot{fbF} &= - (a_1 \dot{b}_1 \dot{A} + a_2 \dot{b}_2 \dot{A} + b_1 \dot{b}_1 \dot{B} + b_2 \dot{b}_2 \dot{B} \\ &\quad + c_1 \dot{b}_1 \dot{C} + c_2 \dot{b}_2 \dot{C} + d_1 \dot{b}_1 \dot{D} + d_2 \dot{b}_2 \dot{D} \\ &\quad + e_1 \dot{b}_1 \dot{E} + e_2 \dot{b}_2 \dot{E} + f_1 \dot{b}_1 \dot{F} + f_2 \dot{b}_2 \dot{F}) \\ &\quad + \text{IF2} \end{aligned}$$

$$\begin{aligned} \ddot{acA} + \ddot{bcB} + \ddot{ccC} + \ddot{dcD} + \ddot{ecE} + \ddot{fcF} &= - (a_1 \dot{c}_1 \dot{A} + a_2 \dot{c}_2 \dot{A} + b_1 \dot{c}_1 \dot{B} + b_2 \dot{c}_2 \dot{B} \\ &\quad + c_1 \dot{c}_1 \dot{C} + c_2 \dot{c}_2 \dot{C} + d_1 \dot{c}_1 \dot{D} + d_2 \dot{c}_2 \dot{D} \\ &\quad + e_1 \dot{c}_1 \dot{E} + e_2 \dot{c}_2 \dot{E} + f_1 \dot{c}_1 \dot{F} + f_2 \dot{c}_2 \dot{F}) \\ &\quad + \text{IF3} \end{aligned}$$

$$\begin{aligned} \ddot{adA} + \ddot{bdB} + \ddot{cdC} + \ddot{ddD} + \ddot{edE} + \ddot{fdF} &= - (a_1 \dot{d}_1 \dot{A} + a_2 \dot{d}_2 \dot{A} + b_1 \dot{d}_1 \dot{B} + b_2 \dot{d}_2 \dot{B} \\ &\quad + c_1 \dot{d}_1 \dot{C} + c_2 \dot{d}_2 \dot{C} + d_1 \dot{d}_1 \dot{D} + d_2 \dot{d}_2 \dot{D} \\ &\quad + e_1 \dot{d}_1 \dot{E} + e_2 \dot{d}_2 \dot{E} + f_1 \dot{d}_1 \dot{F} + f_2 \dot{d}_2 \dot{F}) \\ &\quad + \text{IF4} \end{aligned}$$

$$\begin{aligned} \ddot{aeA} + \ddot{beB} + \ddot{ceC} + \ddot{deD} + \ddot{eeE} + \ddot{feF} &= - (a_1 \dot{e}_1 \dot{A} + a_2 \dot{e}_2 \dot{A} + b_1 \dot{e}_1 \dot{B} + b_2 \dot{e}_2 \dot{B} \\ &\quad + c_1 \dot{e}_1 \dot{C} + c_2 \dot{e}_2 \dot{C} + d_1 \dot{e}_1 \dot{D} + d_2 \dot{e}_2 \dot{D} \\ &\quad + e_1 \dot{e}_1 \dot{E} + e_2 \dot{e}_2 \dot{E} + f_1 \dot{e}_1 \dot{F} + f_2 \dot{e}_2 \dot{F}) \\ &\quad + \text{IF5} \end{aligned}$$



$$\begin{aligned}
a\ddot{f}_1\ddot{A} + b\ddot{f}_2\ddot{B} + c\ddot{f}_3\ddot{C} + d\ddot{f}_4\ddot{D} + e\ddot{f}_5\ddot{E} + f\ddot{f}_6\ddot{F} &= - (a_1\dot{f}_1\ddot{A} + a_2\dot{f}_2\ddot{A} + b_1\dot{f}_1\ddot{B} + b_2\dot{f}_2\ddot{B} \\
&\quad + c_1\dot{f}_1\ddot{C} + c_2\dot{f}_2\ddot{C} + d_1\dot{f}_1\ddot{D} + d_2\dot{f}_2\ddot{D} \\
&\quad + e_1\dot{f}_1\ddot{E} + e_2\dot{f}_2\ddot{E} + f_1\dot{f}_1\ddot{F} + f_2\dot{f}_2\ddot{F}) \\
&\quad + \text{IF6}
\end{aligned}$$

where  $\ddot{A} = \dot{u}$ ,  $\dot{A} = u$ ,  $\ddot{B} = \dot{v}$ ,  $\dot{B} = v$ ,  $\ddot{C} = \dot{w}$ ,  $\dot{C} = w$ ,  $\ddot{D} = \dot{p}$ ,  $\dot{D} = p$   
 $\ddot{E} = \dot{q}$ ,  $\dot{E} = q$ ,  $\ddot{F} = \dot{r}$ ,  $\dot{F} = r$ , terms IF include all non-linear terms such as wave force, wind, rudder deflection---etc.

In the six equations, the right can be set equal to

$I_1$ ,  $I_2$ ,  $\dots$ ,  $I_6$  respectively, thus:

$$I_1 = - (a_1\dot{a}_1\ddot{A} + a_2\dot{a}_2\ddot{A} + b_1\dot{a}_1\ddot{B} + \dots + f_2\dot{a}_2\ddot{F}) + \text{IF1}$$

$$I_2 = - (a_1\dot{b}_1\ddot{A} + a_2\dot{b}_2\ddot{A} + b_1\dot{b}_1\ddot{B} + \dots + f_2\dot{b}_2\ddot{F}) + \text{IF2}$$

$$I_3 = - (a_1\dot{c}_1\ddot{A} + a_2\dot{c}_2\ddot{A} + b_1\dot{c}_1\ddot{B} + \dots + f_2\dot{c}_2\ddot{F}) + \text{IF3}$$

$$I_4 = - (a_1\dot{d}_1\ddot{A} + a_2\dot{d}_2\ddot{A} + b_1\dot{d}_1\ddot{B} + \dots + f_2\dot{d}_2\ddot{F}) + \text{IF4}$$

$$I_5 = - (a_1\dot{e}_1\ddot{A} + a_2\dot{e}_2\ddot{A} + b_1\dot{e}_1\ddot{B} + \dots + f_2\dot{e}_2\ddot{F}) + \text{IF5}$$

$$I_6 = - (a_1\dot{f}_1\ddot{A} + a_2\dot{f}_2\ddot{A} + b_1\dot{f}_1\ddot{B} + \dots + f_2\dot{f}_2\ddot{F}) + \text{IF6}$$

the equations then have the form that follows,

$$aa\ddot{A} + ba\ddot{B} + ca\ddot{C} + da\ddot{D} + ea\ddot{E} + fa\ddot{F} = I_1$$

$$ab\ddot{A} + bb\ddot{B} + cb\ddot{C} + db\ddot{D} + eb\ddot{E} + fb\ddot{F} = I_2$$

$$ac\ddot{A} + bc\ddot{B} + cc\ddot{C} + dc\ddot{D} + ec\ddot{E} + fc\ddot{F} = I_3$$

$$ad\ddot{A} + bd\ddot{B} + cd\ddot{C} + dd\ddot{D} + ed\ddot{E} + fd\ddot{F} = I_4$$

$$ae\ddot{A} + be\ddot{B} + ce\ddot{C} + de\ddot{D} + ee\ddot{E} + fe\ddot{F} = I_5$$

$$af\ddot{A} + bf\ddot{B} + cf\ddot{C} + df\ddot{D} + ef\ddot{E} + ff\ddot{F} = I_6$$



expressing in matrix form:

$$\begin{array}{cccccc|c|c}
 aa & ba & ca & da & ea & fa & \ddot{A} & I_1 \\
 ab & bb & cb & db & eb & fb & \ddot{B} & I_2 \\
 ac & bc & cc & dc & ec & fc & \ddot{C} & I_3 \\
 ad & bd & cd & dd & ed & fd & \ddot{D} & I_4 \\
 ae & be & ce & de & ee & fe & \ddot{E} & I_5 \\
 af & bf & cf & df & ef & ff & \ddot{F} & I_6
 \end{array} = \begin{array}{c} \ddot{A} \\ \ddot{B} \\ \ddot{C} \\ \ddot{D} \\ \ddot{E} \\ \ddot{F} \end{array} = \begin{array}{c} I_1 \\ I_2 \\ I_3 \\ I_4 \\ I_5 \\ I_6 \end{array}$$

Apply Cramer's rule to solve for  $\ddot{A}, \ddot{B}, \ddot{C}, \ddot{D}, \ddot{E}, \ddot{F}$  in terms of  $I_1 \dots I_6$

$$\ddot{A} = \frac{\begin{array}{cccccc|c}
 I_1 & ba & ca & da & ea & fa & I_1 \\
 I_2 & bb & cb & db & eb & fb & I_2 \\
 I_3 & bc & cc & dc & ec & fc & I_3 \\
 I_4 & bd & cd & dd & ed & fd & I_4 \\
 I_5 & be & ce & de & ee & fe & I_5 \\
 I_6 & bf & cf & df & ef & ff & I_6
 \end{array}}{\begin{array}{cccccc|c}
 aa & ba & ca & da & ea & fa & I_1 \\
 ab & bb & cb & db & eb & fb & I_2 \\
 ac & bc & cc & dc & ec & fc & I_3 \\
 ad & bd & cd & dd & ed & fd & I_4 \\
 ae & be & ce & de & ee & fe & I_5 \\
 af & bf & cf & df & ef & ff & I_6
 \end{array}}$$

define the denominator determinant  $\Delta = \Delta$  and for the

nominator let cofactor of  $I_1 \stackrel{\Delta}{=} \text{cof. } aa$

cofactor of  $I_2 \stackrel{\Delta}{=} \text{cof. } ab, \dots$  and cofactor of  $I_6$

$\stackrel{\Delta}{=} \text{cof. } af$  equations becomes:

$$\ddot{A} = \frac{(\text{cof. } aa I_1 + \text{cof. } ab I_2 + \text{cof. } ac I_3 + \text{cof. } ad I_4 + \text{cof. } ae I_5 + \text{cof. } af I_6)}{\Delta}$$



In the same way solve for B, C, -----

$$\therefore B = \frac{(\text{cof } baI_1 \text{ cof } bbI_2 \text{ cof } bcI_3 \text{ cof } bdI_4 \text{ cof } beI_5 \text{ cof } bfI_6)}{\Delta}$$

$$\therefore C = \frac{(\text{cof } caI_1 \text{ cof } cbI_2 \text{ cof } ccI_3 \text{ cof } cdI_4 \text{ cof } ceI_5 \text{ cof } cfI_6)}{\Delta}$$

$$\therefore D = \frac{(\text{cof } daI_1 \text{ cof } dbI_2 \text{ cof } dcI_3 \text{ cof } ddI_4 \text{ cof } deI_5 \text{ cof } dfI_6)}{\Delta}$$

$$\therefore E = \frac{(\text{cof } eaI_1 \text{ cof } ebI_2 \text{ cof } ecI_3 \text{ cof } edI_4 \text{ cof } eeI_5 \text{ cof } efI_6)}{\Delta}$$

$$\therefore F = \frac{(\text{cof } faI_1 \text{ cof } fbI_2 \text{ cof } fcI_3 \text{ cof } fdI_4 \text{ cof } feI_5 \text{ cof } ffI_6)}{\Delta}$$

Then the value of A, A, B, B ----- F, F by integration such that

$$\dot{A} = \frac{dA}{dt} = \int \frac{d^2 A}{dt^2}, \quad A = \int \frac{dA}{dt}$$

A block diagram to compute all of the variables in the set of equations is presented in Fig. 2.

In the computer program that is used for simulation all six equations for six degrees of freedom are used, but are interested in less than six degrees of freedom. The same program can be used by setting the coupling terms of non-used equations equal to zero and one in terms of principal diagonal, e.g. only three degrees of freedom are used in this study, surge, sway and yaw, then all coupling terms



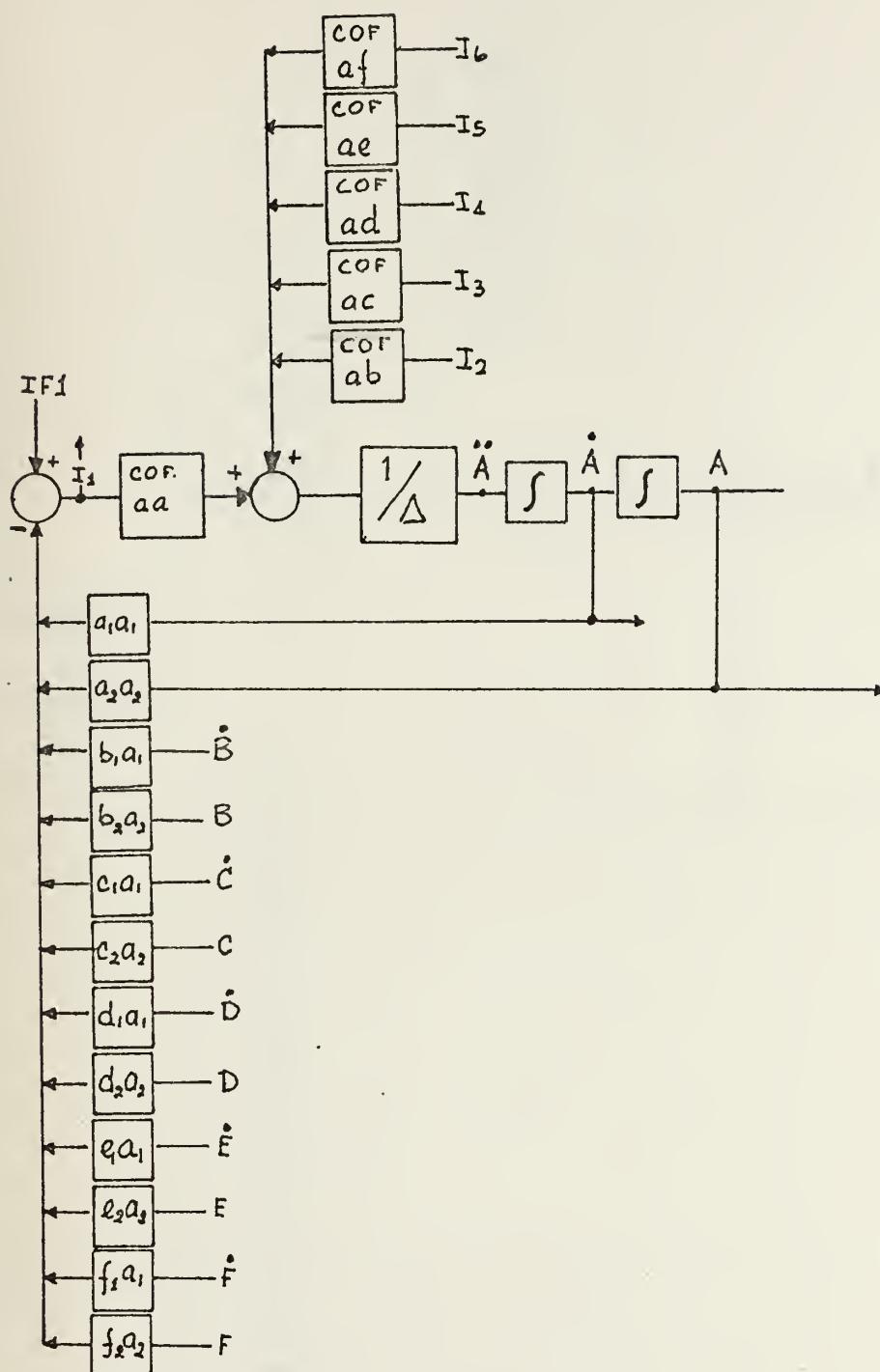


Fig. 2

TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREES OF FREEDOM(Eqn.A)



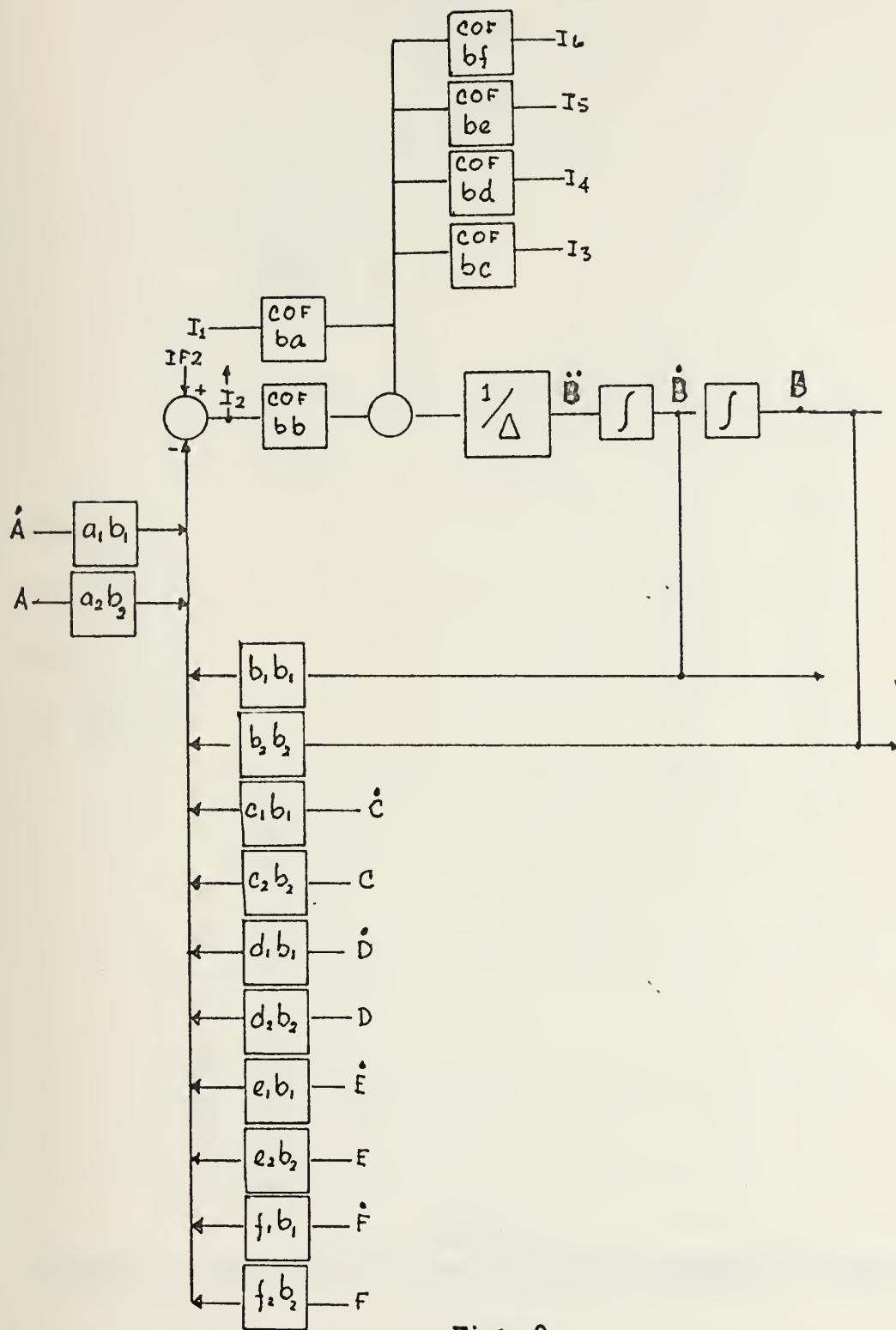


Fig. 2

TRANSEER FUNCTION BLOCK DIAGRAM SIX DEGREES OF FREEDOM(Eqn.B)



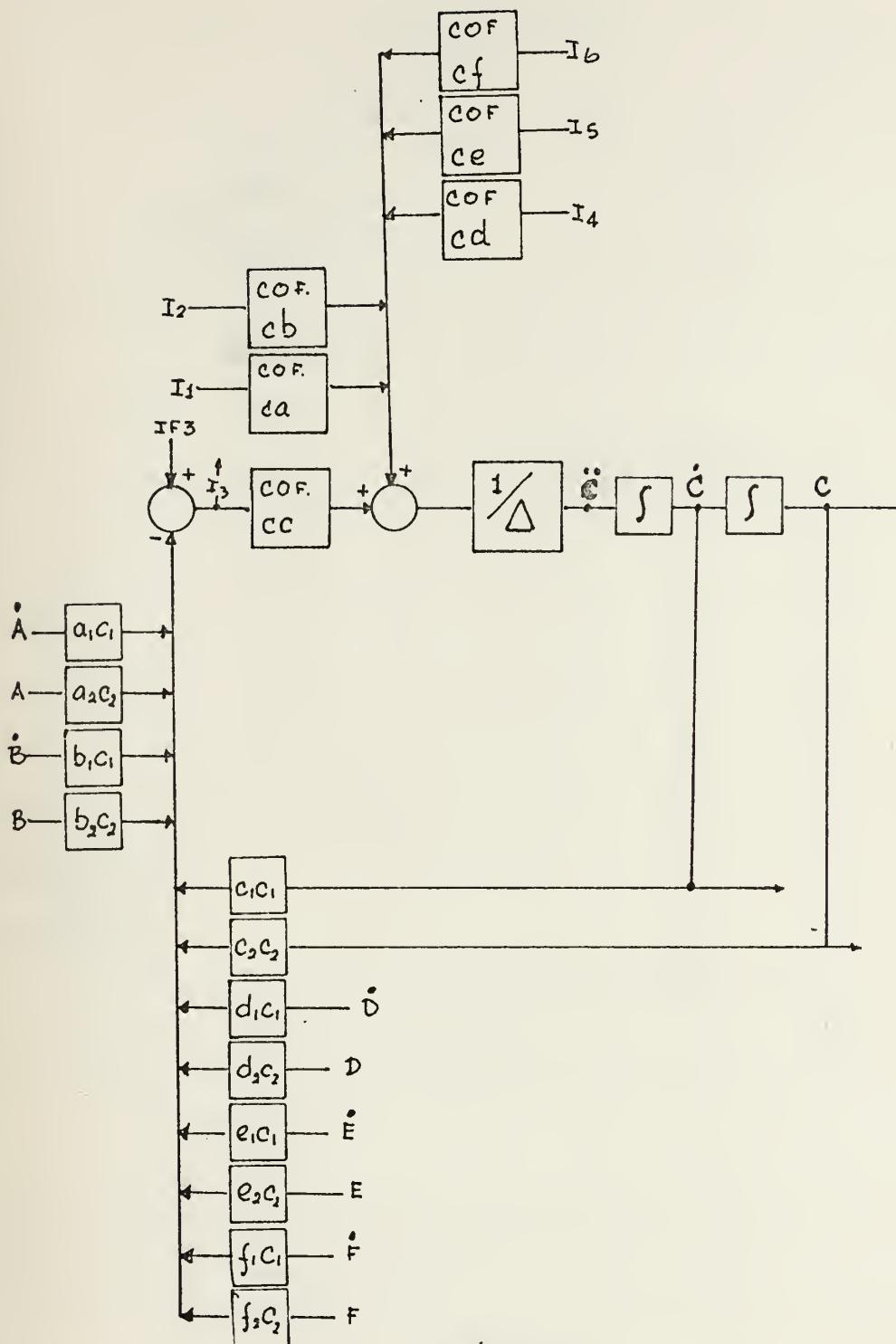


Fig. 2  
TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREES OF FREEDOM(Eqn.C)



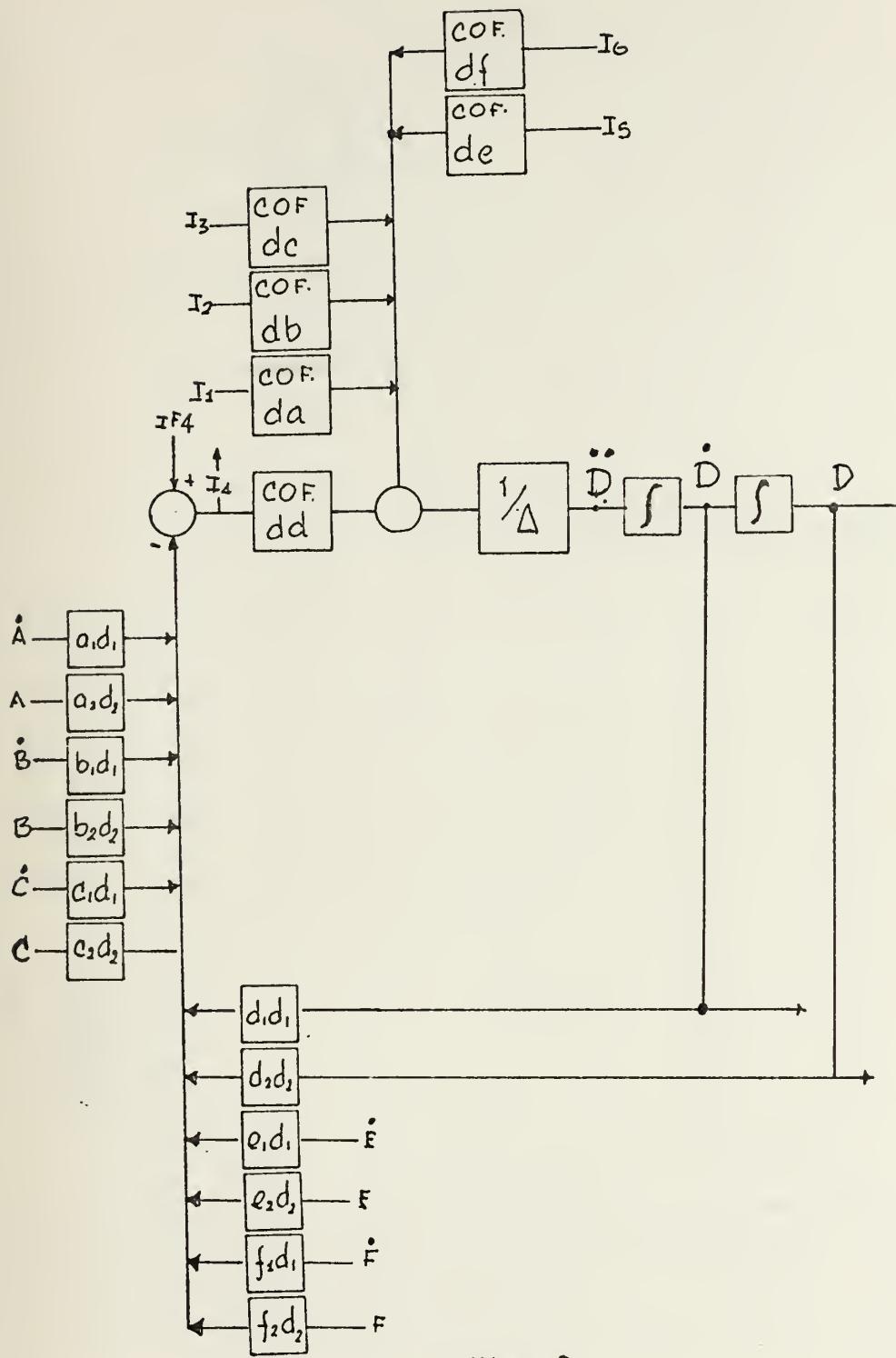


Fig. 2

TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREE OF FREEDOM(Eqn.D)



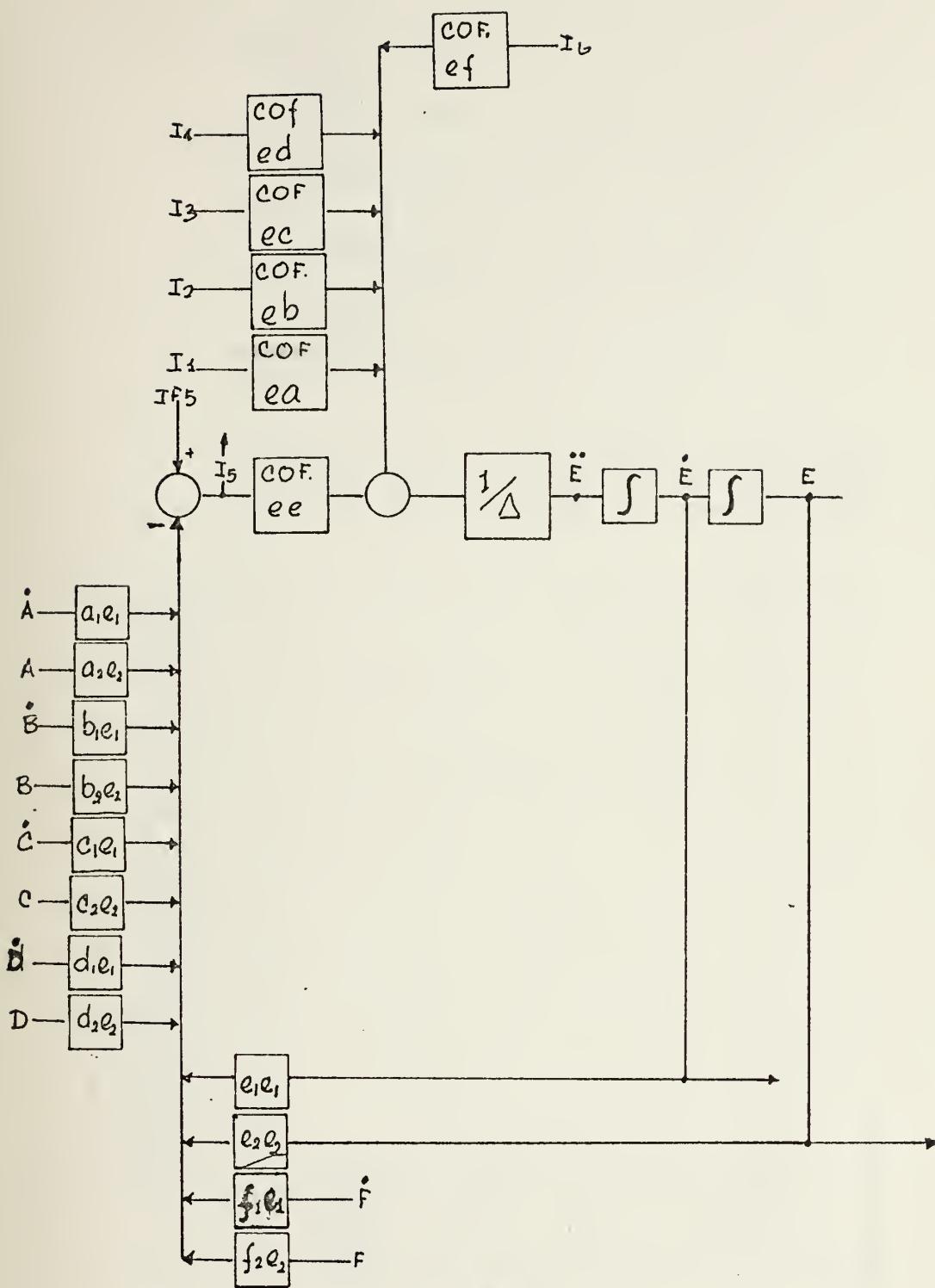


Fig. 2

TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREES OF FREEDOM(Eqn.E)



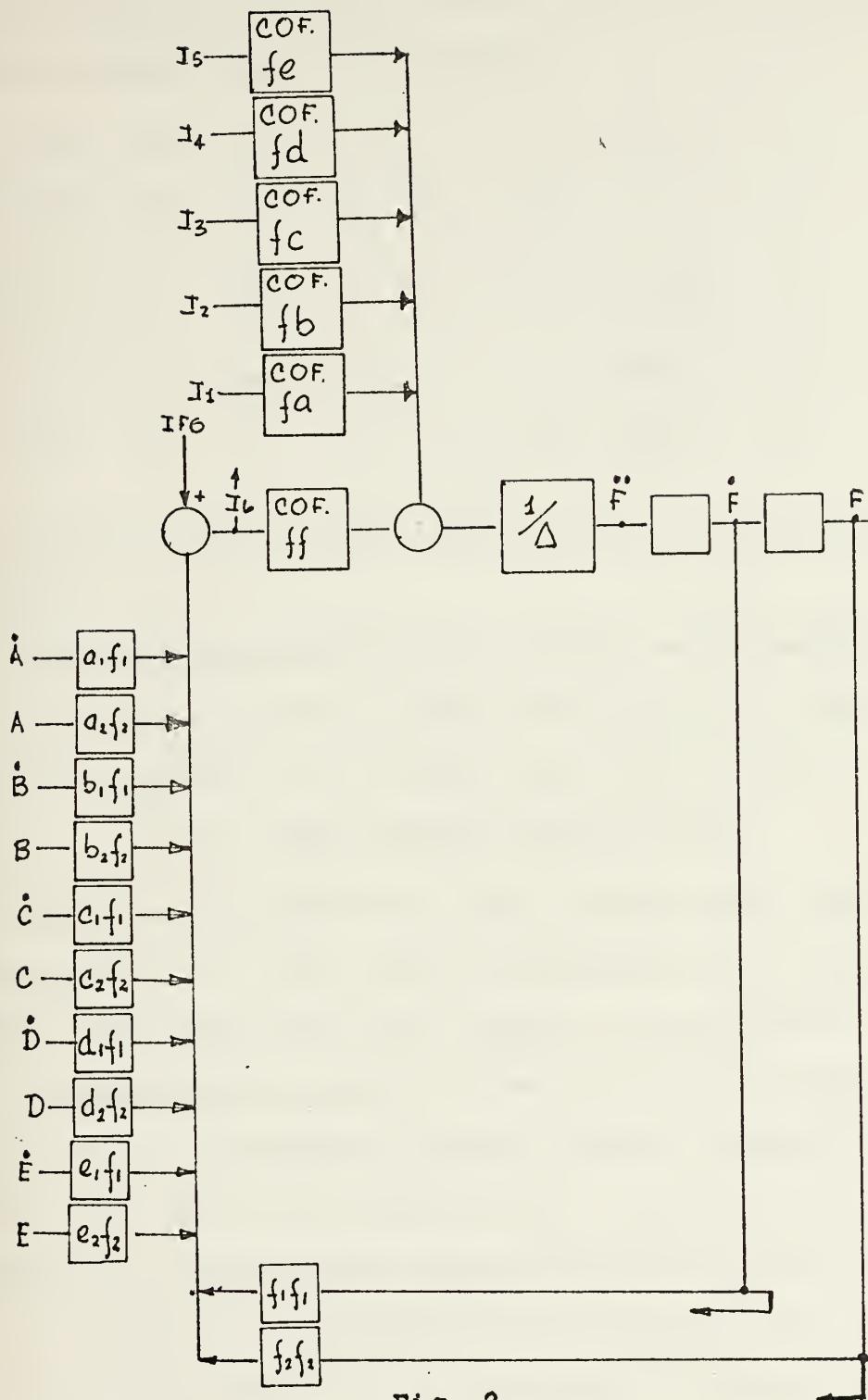


Fig. 2

TRANSFER FUNCTION BLOCK DIAGRAM SIX DEGREES OF FREEDOM (Eqn. F)



are set equal to zero and unused terms on the principal diagonal equal to one, for example :

$$\left| \begin{array}{cccccc} aa & ba & 0 & 0 & 0 & fa \\ ab & bb & 0 & 0 & 0 & fb \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ af & bf & 0 & 0 & 0 & ff \end{array} \right| = \left| \begin{array}{c} \ddot{A} \\ \ddot{B} \\ \ddot{C} \\ \ddot{D} \\ \ddot{E} \\ \ddot{F} \end{array} \right| = \left| \begin{array}{c} I_1 \\ I_2 \\ I_3 \\ I_4 \\ I_5 \\ I_6 \end{array} \right|$$

and the left side of the unused equations are set equal to zero.

With this program non-linear terms can be added such as, rudder deflection of waves and wind, etc., which will be done by adding all of these whose sum is  $IF_N$  e.g.

$$IF_1 = KA_1 \times Dr + KA_2 \times Ds + KA_3 \times Db + NA$$

where Dr, Ds and Db are rudder deflection, canard deflection..... etc. NA is the sum of all non-linear terms that effect the surge equation (X equation).

The program that will be used for solving these equations is the "Continuous Systems Modeling Program" (CSMP) [Ref. 3] in which all constants are declared in the first section and then set the value of matrix for aa, ab, ac ..... and so on (in program AAA is used for aa, AAB for ab ..... AFF for ff). In the initial section values of the COFACTORS aa, ba....are determined. All of the COFACTORS and the subprogram VALUE is used to compute. This subprogram finds the value of the determinant of the



matrix. For all of the COFACTOR terms the element is set equal to one and the rest of the elements in that row and column are set equal to zero. For example, to find the value of COF.aa the following array is obtained:

$$\text{COF aa} = \begin{vmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & bb & cb & db & eb & fb \\ 0 & bc & cc & dc & ec & fc \\ 0 & bd & cd & dd & ed & fd \\ 0 & be & ce & de & ee & fe \\ 0 & bf & cf & df & ef & ff \end{vmatrix}$$

(In the computer program BAA is used for  $a_1 a_1$ , GAA for  $a_2 a_2 \dots$ ). After the value of  $\Delta$  and all cofactors are determined, the dynamic section is used to determine BAA, BAB .....GAA, GAB ..... (if those terms contain variables).

In the dynamic section all variables that are functions of time are determined. The defining relations of the variables are also included in the dynamics section, i.e. UDOT = ADDOT (U=Ā), U=ADOT.....etc. XH,YH,ZH are determined and are the vector terms, X,Y,Z whose origin is fixed on the earth (relative to the earth).



### III. STUDY OF SHIP "D" PERFORMANCE

In this section the computer will be used to solve the equation of motion describing ship "D". The hydrodynamic coefficients and constants that were obtained from NSRDC (NAVAL SHIP RESEARCH AND DEVELOPMENT CENTER) [Ref. 6] for this study will concern only three degrees of freedom such that SURGE, SWAY, and YAW.

Equations of motion of the ship (nondimensional)

#### Axial Force

$$\begin{aligned}
 m(\dot{u} - vr + wg) = & \frac{\rho}{2} \ell (x_{gg} g^2 + x_{rr} r^2 + x_{rp} rp) \\
 & + \frac{\rho}{2} (x_u \dot{u} + x_{vr} vr + x_{wg} wg) \\
 & + \frac{\rho}{2\ell} (x_{vv} v^2) \\
 & + \frac{\rho}{2\ell} u^2 (x_{\delta r \delta r} \delta r^2 + x_{\delta s \delta s} \delta s^2 + x_{\delta b \delta b} \delta b^2) \\
 & + \frac{\rho}{2\ell} (A_1 u^2 + A_2 u \cdot u_c + A_3 u_c^2)
 \end{aligned}$$

#### Lateral Force

$$\begin{aligned}
 m(\dot{v} + ur - wp) = & \frac{\rho}{2} \ell (y_p \dot{p} + y_r \dot{r} + y_{pq} pq) \\
 & + \frac{\rho}{2} (y_{wp} wp + y_v |r| v |r| + u y_r r + y_v \dot{v} + u y_p p) \\
 & + \frac{\rho}{2} (u y_v v + y_{vv} vv + y_v |v| v |v|) \\
 & + \frac{\rho}{2\ell} u^2 (y_{\delta r} \delta r)
 \end{aligned}$$

#### Yawing Moment

$$\begin{aligned}
 I_z \dot{r} + (I_y - I_x) pq = & \frac{\rho}{2} (N_r \dot{r} + N_p \dot{p} + N_{pq} pq) \\
 & + \frac{\rho}{2\ell} (N_v \dot{v} + u N_p p + u N_r r + N_{wp} wp + N_v |v| r)
 \end{aligned}$$



$$+ \frac{\rho}{2\ell^2} (u N_v v + N_{vv} v v + N_v |v|_v |v|_v)$$

$$+ \frac{\rho}{2\ell^2} u^2 (N_{\delta r} \delta r)$$

$\rho$ , the density of fluid is taken as 2 and the terms including w, p. q (heave, roll and pitch) are set equal to zero.

The nonlinear terms such as the squared terms and product terms of v and r are omitted initially. This is in agreement with the small perturbation theory.

After rearranging, the equations become

$$(x_{\dot{u}} - m) \ddot{u} = -x_{\delta r \delta r} \delta r^2 \frac{u^2}{\ell}$$

$$(y_{\dot{v}} - m) \ddot{v} + y_{\dot{r}} \ddot{r} = -\frac{u}{\ell} y_v v - u y_r r - \frac{u^2}{\ell} y_{\delta r} \delta r$$

$$\frac{N_{\dot{v}} \ddot{v}}{\ell} + (N_{\dot{r}} - I_z) \ddot{r} = -\frac{u}{\ell^2} N_v v - \frac{u}{\ell^2} N_r r - \frac{u^2}{\ell^2} N_{\delta r} \delta r$$

Set the left side of the equations equal to  $I_1, \dots, I_6$ , then the matrix equation becomes

$$\left| \begin{array}{cccccc} (x_{\dot{u}} - m) & 0 & 0 & 0 & 0 & 0 \\ 0 & (y_{\dot{v}} - m) & 0 & 0 & 0 & y_r \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & N_{\dot{v}}/\ell & 0 & 0 & 0 & (N_{\dot{r}} - I_z) \end{array} \right| \left| \begin{array}{c} \ddot{u} \\ \ddot{v} \\ \ddot{r} \\ \ddot{D} \\ \ddot{E} \\ \ddot{F} \end{array} \right| = \left| \begin{array}{c} I_1 \\ I_2 \\ I_3 \\ I_4 \\ I_5 \\ I_6 \end{array} \right|$$

where  $\ddot{u} = \ddot{A}$ ,  $\ddot{v} = \ddot{B}$ ,  $\ddot{r} = \ddot{F}$ .



Also set the right side of the equations equal to  $I_1$

-----  $I_6$

$$\begin{array}{c|c|c|c|c|c|c|c} I_1 & 0 & 0 & 0 & 0 & 0 & 0 & A \\ I_2 & 0 & \frac{u}{\ell} Y_v & 0 & 0 & 0 & u Y_r & B \\ I_3 & 0 & 0 & 0 & 0 & 0 & 0 & C \\ I_4 & 0 & 0 & 0 & 0 & 0 & 0 & D \\ I_5 & 0 & 0 & 0 & 0 & 0 & 0 & E \\ I_6 & 0 & \frac{u}{\ell^2} N_v & 0 & 0 & 0 & \frac{u}{\ell} N_r & F \end{array}$$

$$+ \begin{array}{c|c|c|c|c|c|c|c} & A & B & C & D & E & F & \\ & IF1 & IF2 & IF3 & IF4 & IF5 & IF6 & \\ \text{All Zero} & & & & & & & \end{array}$$

$$\text{where } u = \dot{A}, v = \dot{B}, r = \dot{F}, \text{ and } IF1 = -X_{\delta r \delta r} \delta r^2 \frac{u^2}{\ell}$$

$$IF2 = -Y_{\delta r \delta r} \delta r \frac{u^2}{\ell}$$

$$IF3 = 0$$

$$IF4 = 0$$

$$IF5 = 0$$

$$IF6 = -N_{\delta r \delta r} \delta r \frac{u^2}{\ell^2}$$



A. NONLINEAR TERMS NOT INCLUDED

TABLE I

Hydrodynamic Coefficients and Constants of Ship "D"  
for Linear Terms (Non-dimensional)

$$m = 0.0045$$

$$Iz = 0.0003$$

$$Nr = 0.0012$$

$$Nr = -0.0002$$

$$Nv = -0.0012$$

$$Nv = -0.0001$$

$$Xu = -0.00036$$

$$Yv = -0.0025$$

$$Yv = -0.0063$$

$$Yr = 0.004$$

$$X_{\delta r \delta r} = -0.0011$$

$$Y_{\delta r} = 0.0019$$

$$N_{\delta r} = -0.00084$$

$\delta s$  and  $\delta b$  equal zero

In the computer program, set all coefficients in section 1 and set AAA = Xu-m , AAB = 0 ---- AFF = Nr-Iz . After this, use subprogram value to find the determinant and coefficient of AA, BB ---- and then set BBB =  $\frac{u}{l} Yv$  , BFB = uYr ----- BFF =  $\frac{u}{l} Nr$  .

$$IF1 = KA1 \delta r \text{ where } KA1 = -X_{\delta r \delta r} \delta r \frac{u^2}{l}$$

$$IF2 = KB1 \delta r \text{ where } KB2 = -Y_{\delta r} \frac{u^2}{l}$$

$$IF3 = KF1 \delta r \text{ where } KF1 = -N_{\delta r} \frac{u^2}{l^2}$$



Following is the program that used CSMP to determine the turning circles for rudder angles of  $15^\circ$  (-0.2619 rad.),  $25^\circ$  (-0.4365 rad.) and  $35^\circ$  (-0.6111 rad.). Result of this study is presented in Fig. 3. The turning rate as a function of time is interested and the results of this analysis are presented in Fig. 4. The ships turning performance expressed in transfer ship lengths as time is shown in Fig. 5 and the heading angle as a function of time is given in Fig. 6. Fig. 7 shows the results of the zig-zag maneuver, curve shown the yaw angle and rudder angle in degree as a function of time, for this study the same program was used, but set DR in dynamic section:

```
DR= -0.06984 (RAMP(0.0)-RAMP(5.0))+0.04656 (RAMP(40.0)...
    -RAMP(65.0)-RAMP(145.0)+RAMP(170.0)+RAMP(250.0)....
    -RAMP(265.0)-RAMP(345.0)+RAMP(425.0)+RAMP(440.0))
```

and use prepare statement prepare X, YAWD and prepare X, DOO (YAWD = YAW\* 57.273, DOO = DR\* 57.273).











```

AEC=0 0
ADE=1 0
ADF=0 0
AEE=0 0
AEB=0 0
AED=0 0
AEF=0 0
AFB=L C* YR DCT
AFC=0 0
AFC=0 0
AFC=L R DCT-12
SECTION 2A
CCCFAA=VALUE( 'AAA', 0, 0 )
CCCFAB=VALUE( 'AAA', 1, 1 )
CCCFAC=VALUE( 'AAA', 2, 1 )
CCCFAD=VALUE( 'AAA', 3, 1 )
CCCFAE=VALUE( 'AAA', 4, 1 )
CCCFAF=VALUE( 'AAA', 5, 1 )
CCCFBA=VALUE( 'AAA', 6, 1 )
CCCFBB=VALUE( 'AAA', 1, 2 )
CCCFBC=VALUE( 'AAA', 2, 2 )
CCCFBD=VALUE( 'AAA', 3, 2 )
CCCFBE=VALUE( 'AAA', 4, 2 )
CCCFBF=VALUE( 'AAA', 5, 2 )
CCCFCA=VALUE( 'AAA', 6, 2 )
CCCFCB=VALUE( 'AAA', 1, 3 )
CCCFCC=VALUE( 'AAA', 2, 3 )
CCCFCE=VALUE( 'AAA', 3, 3 )
CCCFCD=VALUE( 'AAA', 4, 3 )
CCCFDA=VALUE( 'AAA', 5, 3 )
CCCFDB=VALUE( 'AAA', 1, 4 )
CCCFCC=VALUE( 'AAA', 2, 4 )
CCCFDD=VALUE( 'AAA', 3, 4 )
CCCFDE=VALUE( 'AAA', 4, 4 )
CCCFDA=VALUE( 'AAA', 5, 4 )
CCCFDB=VALUE( 'AAA', 6, 4 )
CCCFEA=VALUE( 'AAA', 1, 5 )
CCCFEB=VALUE( 'AAA', 2, 5 )
CCCFEC=VALUE( 'AAA', 3, 5 )
CCCFED=VALUE( 'AAA', 4, 5 )
CCCFEE=VALUE( 'AAA', 5, 5 )

```



```

CCFFFA=VALUE(AAA,1,6)
CCFFFB=VALUE(AAA,2,6)
CCFFFC=VALUE(AAA,3,6)
CCFFFD=VALUE(AAA,4,6)
CCFFFE=VALUE(AAA,5,6)
CCFFF=VALUE(AAA,6,6)

```

DYNAMIC

DCC=-DR\*57.273

```

X=TIME
KA1=-XDRDR*U*U*DR/LC
KB1=-YDR*U*U*U/LC
KC1=-KDR*U*U/LC**2
KF1=-NDR*U*U/LC**2
BDB=U*YV/LC
BEC=U*KV/LC**2
B2F=U*NV/LC**2
BCB=U*YP
BCC=U*KPC/LC
BCF=U*NR/LC
BFE=U*YR
BFD=U*KR/LC
BFF=U*NR/LC

```

\*SECTION 3-DEFINITIONS

```

L=ADOT=ADDOT
V=ECOT
W=COT=CUDCT
N=CDOT=DDOT
P=CDOT=DDOT
G=CCT=EDCCT
G=EDOT=EDCCT
R=FDOT=FDCT
D1=DR
C2=CS
C3=CB

```

```

DABR=ABS(R)
AEV=ABS(V)
AEG=ABS(G)
AEW=ABS(W)
AEF=ABS(P)

```

\*KINETIC RELATICS

```

RUDOT=P+YADOT*SIN(PITCH)
PUDCT=G*COS(ROLL)-R*SIN(ROLL)
YACCT=(K+PIDOT*SIN(ROLL))/COS(PITCH)*COS(ROLL)
YAWRD=YADOT*57.273

```



RCULL=INTGRL(0.,RODCT)

YAW=C=YAW\*57.273 - V\*SIN(YAW)

XFDCOT=U\*SIN(YAW)+V\*CCS(YAW)

2YVDCOT=-U\*SIN(PITCH)+W\*CCS(PITCH)

\*SECTION 4 - PROGRAMMED SYMULATION

11=-BA\*A-GOT-GDA\*D-BEA\*BDOT-GAA\*A-BBA\*BDOT-GBA\*E-BCA\*CDC-T-GCA\*C\*•

11=-ECA\*DDOT-GDA\*D-BEA\*EDOT-GEA\*E-BFA\*FDOT-GFA\*F+F+IF1

12=-EAC\*ADOT-GDA\*D-BEA\*EDOT-GEA\*E-BFA\*FDOT-GFA\*F+F+IF2

13=-BAC\*DDOT-GDA\*D-BEA\*EDOT-GEA\*E-BFA\*FDOT-GFA\*F+F+IF3

13=-ELC\*DDOT-GDA\*D-BEA\*EDOT-GEA\*E-BFA\*FDOT-GFA\*F+F+IF4

14=-BAD\*DDOT-GDA\*D-BEA\*EDOT-GEA\*E-BFA\*FDOT-GFA\*F+F+IF5

14=-ECD\*DDE\*ADOT-GDA\*D-BEA\*EDOT-GEA\*E-BFA\*FDOT-GFA\*F+F+IF6

15=-BAE\*DDOT-GDA\*D-BEA\*EDOT-GEA\*E-BFA\*FDOT-GFA\*F+F+IF7

15=-ECE\*DDOT-GDA\*D-BEA\*EDOT-GEA\*E-BFA\*FDOT-GFA\*F+F+IF8

1c=-BAF\*ALOT-GDA\*D-BEA\*EDOT-GEA\*E-BFA\*FDOT-GFA\*F+F+IF9

-BDF\*KAI\*D1+KA2\*D2+KA3\*D3+NA

1F2=KB1\*D1+KB2\*D2+KB3\*D3+NC

1F3=KC1\*D1+KC2\*D2+KC3\*D3+ND

1F4=KD1\*L1+KD2\*D2+KD3\*D3+NE

1F5=KE1\*D1+KE2\*D2+KE3\*D3+NF

ACDOT=(COFAA\*11+COFAB\*12+COFAC\*13+COFAD\*14+COFAE\*15+COFAF\*16)/DEL

BCDOT=(COFBAA\*11+COFBAB\*12+COFBAC\*13+COFBBD\*14+COFBEE\*15+COFBFF\*16)/DEL

CCDOT=(COFCDA\*11+COFCDB\*12+COFCDC\*13+COFCDD\*14+COFCE\*15+COFCEE\*16)/DEL

EDDOT=(COFFEA\*11+COFFEB\*12+COFFEC\*13+COFFED\*14+COFFEE\*15+COFFF\*16)/DEL

FCDOT=(COFFFA\*11+COFFFB\*12+COFFBD\*13+COFFDT)

ALDOT=INTGRL(0.,BDDCT)

CLDOT=INTGRL(0.,CDDCT)

EDOT=INTGRL(0.,DDCT)

FDCT=INTGRL(0.,FDDCT)

A=INTGRL(0.,ADCT)

B=INTGRL(0.,BDCT)

C=INTGRL(0.,CDCT)

D=INTGRL(0.,DDCT)

E=INTGRL(0.,EDOT)



```

F= INTGRL(C*,FDOT)
* SECTION 5-OUTPUT CHARACTERISTICS
  TIMER DELT=0.01,FINTIM=240.0,CUTDEL=1.0,PRDEL=1.0
    PREPARE YH,XH
  END
  PARAN CR=-0.4265
  END
  PARAN CR=-0.6111
  END
  STCP

```

```

CCMN CN  FUNCTION VALUE(Y,I,N)
      DIMENSION X(6,6),Y(6,6)
      DC 1 M1=1,6
      DC 1 M2=1,6
      1  IF((I,1,0,C)=Y(M1,M2))
          GO TO 1CC
      X(1,1)=0.
      X(1,2)=0.
      X(1,3)=C.
      X(1,4)=0.
      X(1,5)=C.
      X(1,6)=0.
      X(2,1)=C.
      X(2,2)=0.
      X(2,3)=C.
      X(2,4)=0.
      X(2,5)=C.
      X(2,6)=0.
      1CC  CCNTINUE
      WRITE(6,49),I,M
      49  FOR50 M1=1,6
      WRITE(6,51)(X(M1,M2),M2=1,6)
      51  FCRTMAT(1C(LX,E13.6))
      CCNTINUE
      N=6
      DC=1.0C
      CC 34 L=1,N
      KF=0
      Z=C*0
      DC 1 Z K=L,N
      1 F((2-ABSS(X(K,L)))/11,12,12
      11  Z=ABSS(X(K,L))
      K=P

```



```

12 CCNTINUE
13 1F(L-KP)13,20,20
14 J=L,N
15 Z=X(L,J)
16 X(KP,J)=X(KP,J)
17 X(KP,J)=Z
18 CC=-DD
19 1F(L-N)31,40,40
20 LP1=L+1
21 CC=34 K=L,P1,N
22 1F(X(K,L))32,34,32
23 RAT10=X(K,L)/X(L,L)
24 CC=33 J=L,P1,N
25 X(K,J)=X(K,J)-RATIC*X(L,J)
26 CCNTINUE
27 CC=41 K=1,N
28 CC=DD*X(K,K)
29 D=DD
30 VALUE=D
31 WRITE(6,52),I,M,VALUE
32 C0F!,II,II,'=' ,E15.6)
33 RETURN
34 END JCB

```



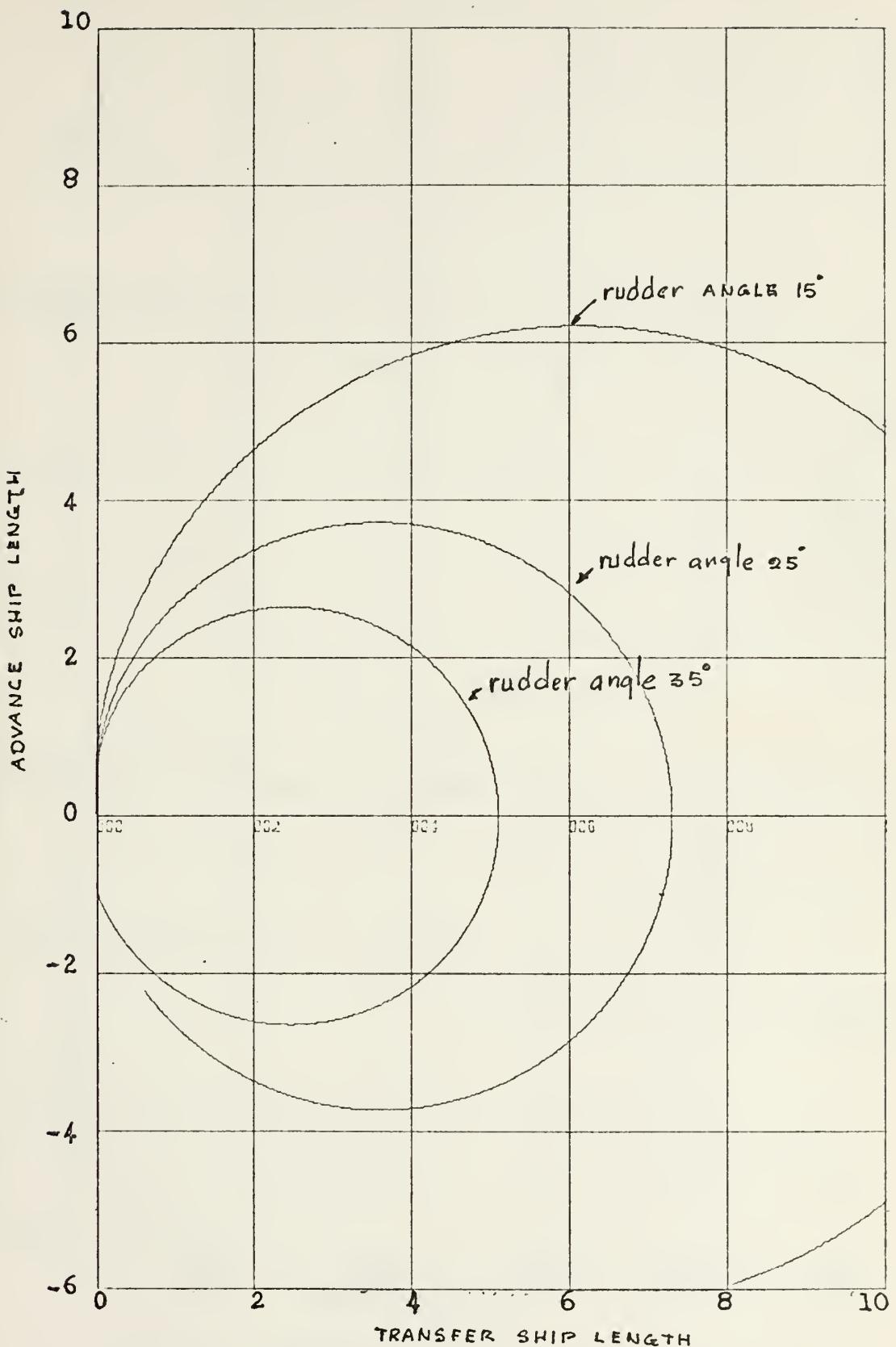


FIG. 3 ADVANCE VS. TRANSFER SHIP LENGTH  
(RUDDER 15°, 25° & 35°)



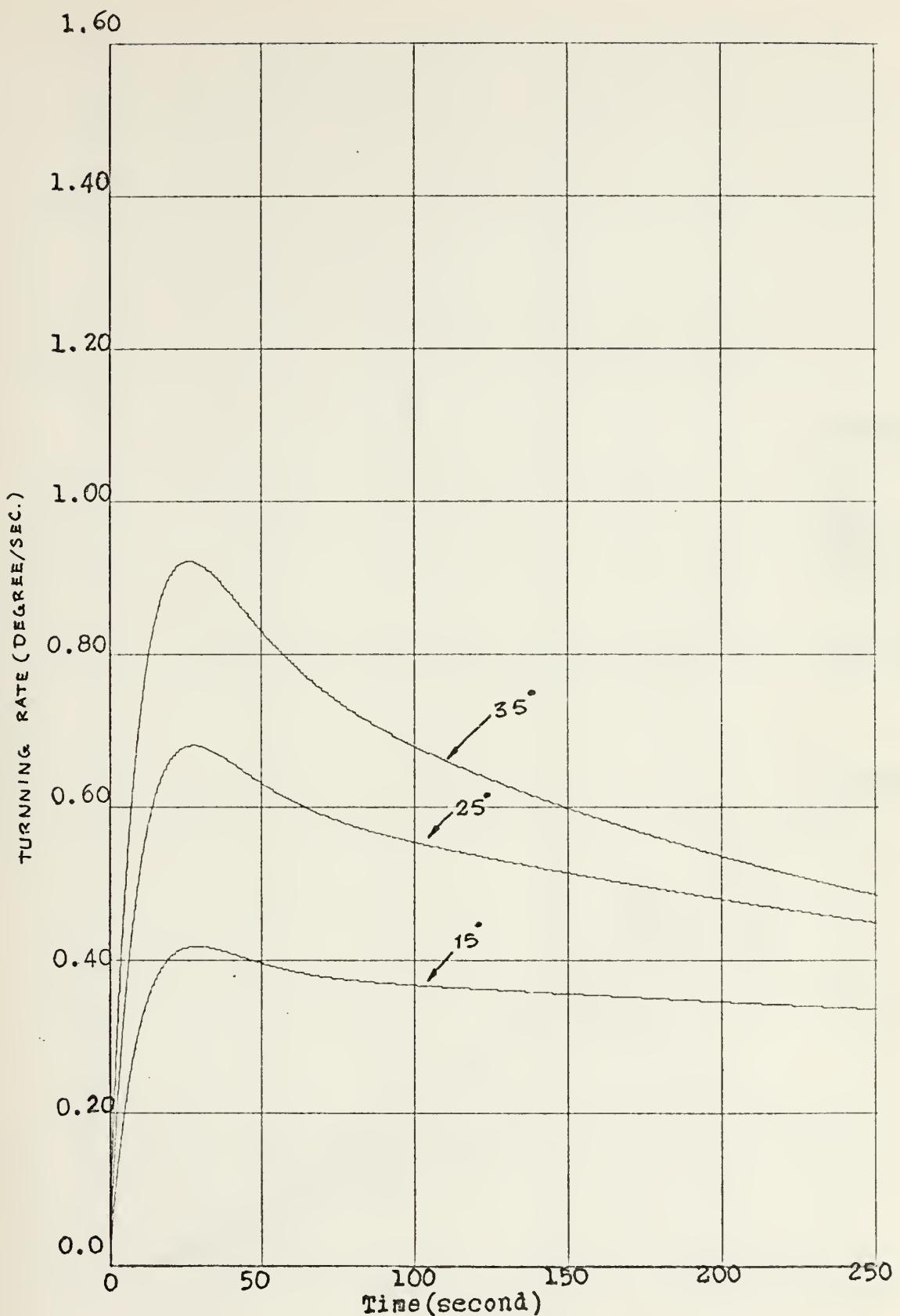


FIG. 4 TURNNING RATE VS. TIME  
(RUDDER 15°, 25°, 35°)



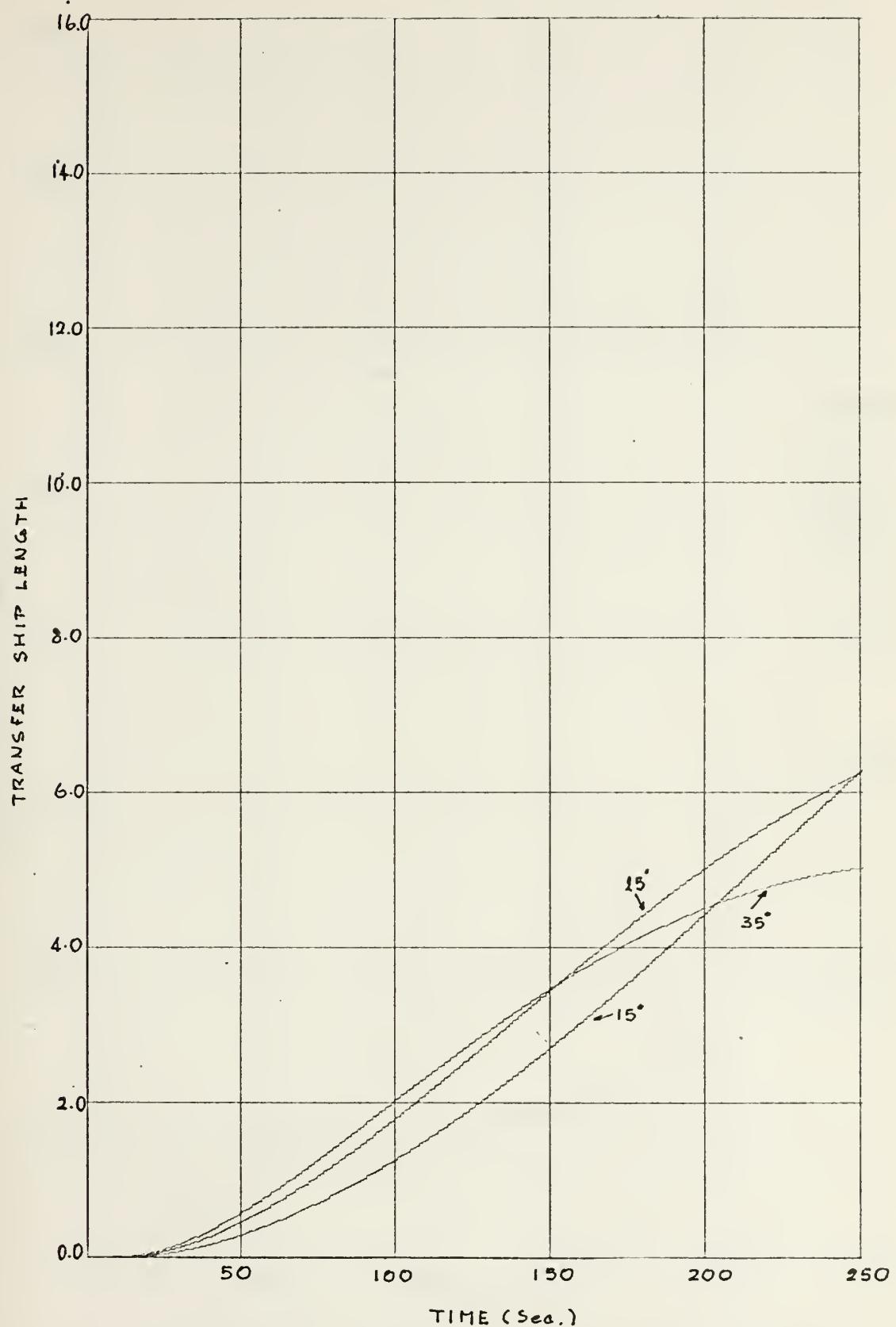


FIG. 5 TRANSFER SHIP LENGTH VS. TIME  
(RUDDER ANGLE 15°, 25°, 35°)



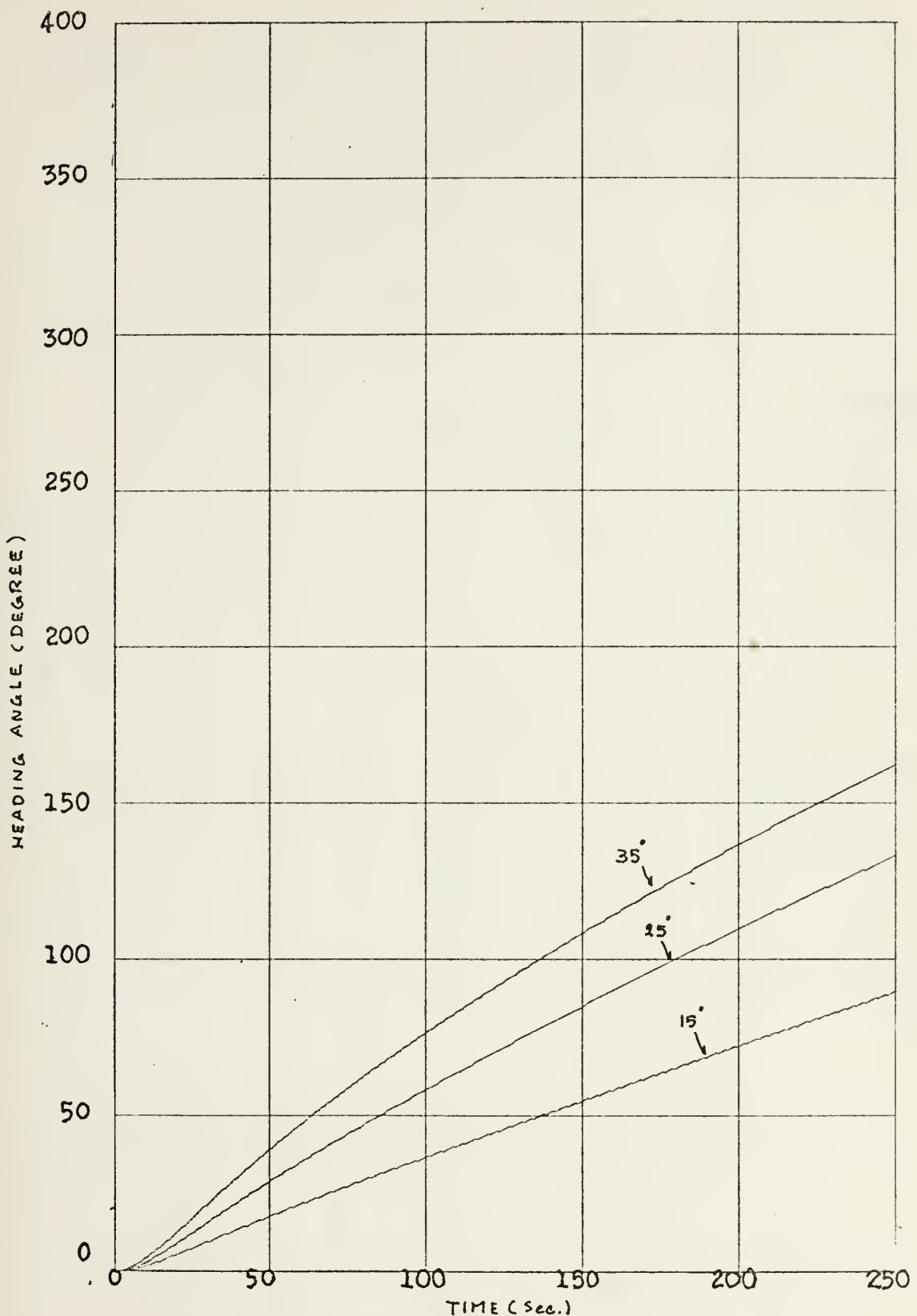


FIG. 6 HEADING ANGLE AS FUNCTION OF TIME  
(RUDDER ANGLE 15°, 25° & 35°)



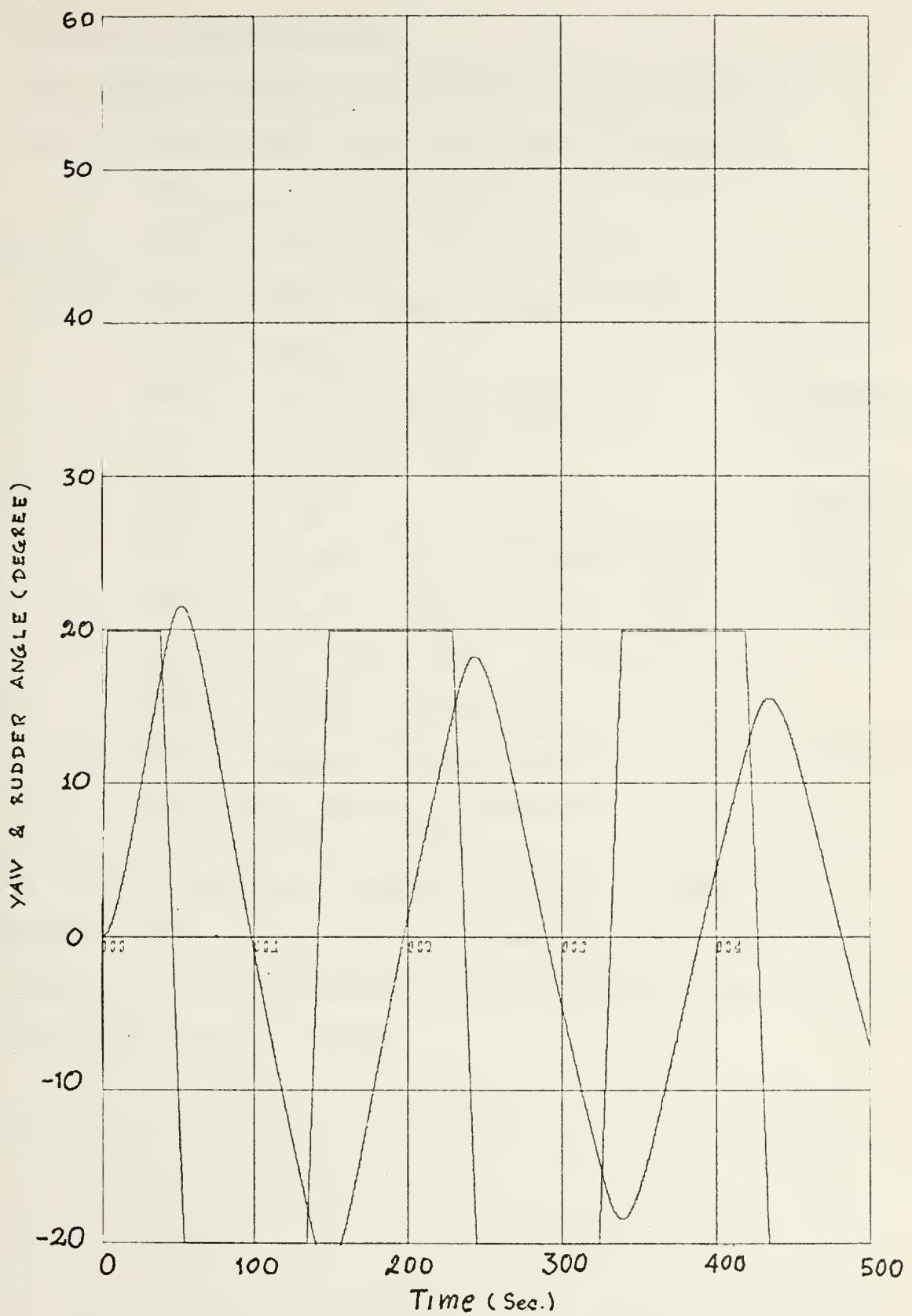


Fig. 7 YAW & RUDDER ANGLE VS. TIME ZIG-ZAG MANOEUVRE



## B. INCLUDED NONLINEAR TERMS

The computer program 2 is the same as the computer program 1, but nonlinear terms are added by setting:

$$NA = NA1 + NA2 + NA3 + NA4$$

where  $NA1 = -1(x_{qq}q^2 + x_{rr}r^2 + x_{rp}rp)$

$$NA2 = -(mvr + x_{vr}vr + x_{wq}wq + mwq)$$

$$NA3 = -(x_{vv}v^2)/1$$

$$NA4 = -(Alu^2 + A2uu_c + A3u_c^2)/1$$

$$NB = NB1 + NB2 + NB3$$

where  $NB1 = -1Y_{pq}pq$

$$NB2 = -(Y_{wp}wp + Y_{v|r}v|r| + mur + mwp)$$

$$NB3 = -(Y_{wv}wv + Y_{|v|v}|v|v)/1$$

$$NF = NF1 + NF2 + NF3$$

where  $NF1 = -N_{pq}pq + (Iy - Ix) pq$

$$NF2 = -(N_{wp}wp + N_{|v|r}|v|r)/1$$

$$NF3 = -(N_{wv}wv + N_{|v|v}|v|v)/1^2$$

Again setting terms that include w, p, and q (heave, roll and pitch) equal to zero, and set number of known terms from Table II into section 1 of CSMP program (unknown coefficients set equal to zero).



TABLE II

## Hydrodynamic Coefficients of Ship "D" for Nonlinear Terms (non-dimensional)

$$X_{rr} = 0.00005$$

$$X_{vr} = 0.00241$$

$$X_{vv} = -0.00341$$

$$Y_{v|v|} = -0.0416$$

$$N_{v|v|} = -0.01002$$

PROPELLION RATIO  $\Delta \eta$ 

$$\eta \geq 0.45 \quad -1.0 \leq \eta < 0.45 \quad \eta \leq 1.0$$

A1	-0.00004	-0.00032	-0.00117
A2	-0.00035	0.00070	-0.00100
A3	0.00099	0	-0.00085

Fig. 8 - Fig. 12 are the same results as Fig. 3 - Fig. 7.

The results of computer program 2 are more accurate than the computer program 1, when compared with free running model test of NSRDC [Ref. 6]. Fig. 13 and Fig. 14 when studying the stability of the ship by applying a force moment to the ship in computer program, set

NFL 0.0001\*(STEP(10.0)-STEP(10.01)) (can use NFL because NFL in this program equal zero)

Study direction stability of the ship by plotting direction of the ship (used advance VS. transfer ship length) and check heading angle of the ship by plotting YAW VS. TIME.



COMPUTER PROGRAM 2 (INCLUDED NON LINEAR TERMS)  
 SURFACE SHIP IN THREE DEGREES OF FREEDOM (SIMULATES THE DYNAMICS OF A SHIP IN 6 DEGREES OF FREEDOM)  
 THIS PROGRAM SIMULATES THE MOTION ARE ASSUMED TO BE IN THE FORM:  

$$\begin{aligned}
 & \text{HAA} * \text{A} + \text{HBA} * \text{B} + \text{HCA} * \text{C} + \text{HDA} * \text{D} + \text{HEA} * \text{E} + \text{HFA} * \text{F} = \text{IFA} \\
 & \text{HAB} * \text{A} + \text{HBB} * \text{B} + \text{HCB} * \text{C} + \text{HDB} * \text{D} + \text{HEB} * \text{E} + \text{HFB} * \text{F} = \text{IFB} \\
 & \text{HAC} * \text{A} + \text{HBC} * \text{B} + \text{HCC} * \text{C} + \text{HDC} * \text{D} + \text{HEC} * \text{E} + \text{HFC} * \text{F} = \text{IFC} \\
 & \text{HAE} * \text{A} + \text{HBD} * \text{B} + \text{HCE} * \text{C} + \text{HDD} * \text{D} + \text{HEE} * \text{E} + \text{HFD} * \text{F} = \text{IFD} \\
 & \text{HAF} * \text{A} + \text{HBE} * \text{B} + \text{HCE} * \text{C} + \text{HDE} * \text{D} + \text{HEF} * \text{E} + \text{HFF} * \text{F} = \text{IFE} \\
 & \text{HBI} * \text{S} * \text{A} + \text{HBIJ} * \text{S} * \text{B} + \text{HCF} * \text{C} + \text{HBF} * \text{D} + \text{HDF} * \text{E} + \text{HFF} * \text{F} = \text{IFF}
 \end{aligned}$$
 WHERE I IS THE COLUMN AND J IS THE ROW  
 HI ARE POLYNOMIALS OF THE FORM:  $\text{AIJ} * \text{S} * \text{A} + \text{BIJ} * \text{S} * \text{B} + \text{C1} * \text{C} + \text{C2} * \text{D} + \text{D1} * \text{E} + \text{D2} * \text{F}$   
 A, B, C, D, E, F ARE THE VARIABLES  
 I, J = K1, K2, K3, K4, K5, K6, K7, K8  
 MUST BE DEFINED IN SECTION 2 FROM THE HYDRODYNAMIC  
 COEFFICIENTS AND THE NON DIMENSIONALIZATION PROCESS.  
 THE VARIATIONS FOR AIJ MUST BE DEFINED AS INDICATED IN SECTION 2A  
 IF NOT ALL EQUATIONS ARE USED, THE NON USED AIJ MUST BE SET = 1  
 IN SITUATION CAN ARISE WHEN ONLY THE LATERAL OR LONGITUDINAL DYN. IS DESIRED  
 IN THIS CASE THE ZEROS IN THE SECTION MUST BE DECLARED IN  
 SECTION 2, UP TO A MAXIMUM OF 85. THE EXCESS MUST BE DECLARED IN  
 SECTION 2  
 D1, D2 ARE THE DEFLECTIONS OF RUDDERS, CANARDS ETC  
 NJ ARE TERMS IN WHICH CAN BE INCLUDED THE COMPONENTS OF FORCES AND MOMENTS  
 DUE TO WAVES AND/OR WIND. ALSO CAN LINEAR TERMS CAN BE DEFINED BUT UP TO  
 THE FIRST DERIVATIVE OF THE BASIC VARIABLES (A, B, C, D, E, F). IF OTHER EFFECTS OR  
 RUDDERS, CANARDS ETC. ARE REQUIRED THEY CAN ALSO BE INCLUDED IN IFJ  
 ALL DEFINITIONS CAN BE INCLUDED IN SECTION 3. NO OTHER REQUIRED  
 HYDRODYNAMICS COEFFICIENTS AND OTHER PARAMETERS ARE INTRODUCED IN SECTION 1  
 SECTION 2  
 D1, D2 ARE THE DEFLECTIONS OF RUDDERS, CANARDS ETC  
 NJ ARE TERMS IN WHICH CAN BE INCLUDED THE COMPONENTS OF FORCES AND MOMENTS  
 DUE TO WAVES AND/OR WIND. ALSO CAN LINEAR TERMS CAN BE DEFINED BUT UP TO  
 THE FIRST DERIVATIVE OF THE BASIC VARIABLES (A, B, C, D, E, F). IF OTHER EFFECTS OR  
 RUDDERS, CANARDS ETC. ARE REQUIRED THEY CAN ALSO BE INCLUDED IN IFJ  
 ALL DEFINITIONS CAN BE INCLUDED IN SECTION 3. NO OTHER REQUIRED  
 HYDRODYNAMICS COEFFICIENTS AND OTHER PARAMETERS ARE INTRODUCED IN SECTION 1  
 THE DETERMINANT AND COFACTORS ARE PRINTED; COFOO=DEL, COF11=COFAA, COF21=COFAB...  
 SINCE THEY ARE PART OF THE MAIN SIMULATION  
 FIXED M, M2, M1, K, L, N, LPI, KP, J, I DEFINED AS FIXED CAN NOT BE USED  
 SECTION 1 COEFFICIENTS FOR A PARTICULAR SHIP  
 SURFACE SHIP 3 DEGREE OF FREEDOM  
 INCCAN ACCTC=0.04263  
 PARAN LDC=0.4263  
 PARAN DR=-0.2615  
 PARAN XCRDR=-0.0011  
 PARAN YCR=0.0C19  
 PARAN NCR=-0.00084  
 PARAN XUDCT=-0.00036  
 PARAN ML=C.0045







```

W=CCDOT
P=DDOT
Q=DDOT
R=EDOT
D1=DR
D2=DS
C2=CB
AER=ABS(R)
AEV=ABS(V)
AEG=ABS(Q)
AEW=ABS(W)
AEP=ABS(P)
*KINEMATIC RELATIONS
RCCDT=P+YADOT*SIN(PITCH)
PIDCT=Q*COS(ROLL)-R*SIN(ROLL)
YADCT=(R+PIDCT*SIN(ROLL))/COS(PITCH)*COS(ROLL)
YAWRD=YADCT*57.273
RCLL=INTGRL(0.,RDOCT)
PITCH=INTGRL(0.,YADCT)
YAW=INTGRL(0.,YADCT*57.273)
XHDCY=U*COS((YAW))-V*SIN((YAW))
YHDCY=U*SIN((YAW))+V*COS((YAW))
XVDCY=U*COS((PITCH))+W*SIN((PITCH))
ZVDCY=-L*SIN((PITCH))+W*COS((PITCH))
XHDT=INTGRL(0.,XHDOCT)
YHDT=INTGRL(0.,YHDOCT)
XV=INTGRL(0.,XVDOCT)
ZV=INTGRL(0.,ZVDOCT)
*SECTION 4 - PRGRM NED SYMBOLATION
1=-BAE*ADCT-GAA*BBA*BDOT-GBA*E-BFA*EDOT-GCA*C:::
2=ECB*DDOT-GDA*D-BEA*EDOT-GEA*E-BFA*EDOT-GFA*F+IF1
3=-BDB*DDOT-GAB*D-BEB*BDOT-CBB*F+IF2
4=-BDC*DDOT-GDB*D-BEB*EDOT-GEB*F+IF3
5=-BAC*ADOT-GAC*D-BEC*EDOT-GEC*E-BFC*F+IF4
6=-BDC*DDCT-GDC*D-BEC*EDOT-GDC*E-BFD*F+IF5
7=-BDC*DDCT-GDD*D-BEC*EDOT-GEC*E-BFC*F+IF6
8=-BDC*DDCT-GDE*D-BEE*EDOT-GEE*E-BFE*F+IF7
9=-BDF*DDOT-GDF*D-BEF*EDOT-GEF*E-BFF*F+IF8
*NCN LINEAR RELATIONS
NA=NA1+NA2+NA3+NA4
NB=NB1+NB2+NB3+NB4

```



```

NF=NF1+NF2+NF3
NA1=-LC*(XQC*Q**2+XRR*R**2+XRP*R*R)
NA2=-(ML*V*XVR*V*R*XW*G*W*G-ML*W*G)
NA3=-(XVV*V**2)/LC
NA4=-(A1*U**2+2*A2*U*UC+A3*UC**2)/LC
NB1=-LC*YPC*P*Q
NB2=-(YWV*W*V+Y1V*ABV*V)/LC
NB3=-(YVW*W*V+V1V*ABV*V)/LC
NF1=-NPG*P*Q+(IY-IX)*P*Q
NF2=-(NWP*W*P+NIY*ABV*R)/LC
NF3=-(NWy*W*V+NIY*ABV*V)/LC**2
IF1=KA1*D1+KA2*D2+KA3*D3+NA
IF2=KB1*D1+KB2*D2+KB3*D3+NA
IF3=KC1*D1+KC2*D2+KC3*D3+NC
IF4=KD1*D1+KD2*D2+KD3*D3+ND
IF5=KE1*D1+KE2*D2+KE3*D3+NE
IF6=KF1*D1+KF2*D2+KF3*D3+NF
ACDCT=(COFAA*I1+COFAB*I2+COFAC*I3+COFAD*I4+CCFAE*I5+COFAF*I6)/DEL
BDDCT=(CCFBA*I1+CCFBB*I2+CCFBC*I3+CCFBD*I4+CCFBE*I5+CCFEE*I6)/DEL
CDDCT=(COFCB*A1+COFCB*B1+COFCB*C1+COFCB*D1+COFCB*E1+COFCB*F1)/DEL
ECDCT=(COFDA*I1+COFDB*I2+COFDC*I3+COFDD*I4+COFDE*I5+COFDF*I6)/DEL
FDDCT=(COFFA*I1+COFFB*I2+COFFC*I3+COFFF*I4+COFFE*I5+COFFF*I6)/DEL
ACCT=INTGRL(ADCT,ACCT)
BDCT=INTGRL(BDCT,BDCT)
CDDCT=INTGRL(CDDCT,CDDCT)
DDCT=INTGRL(DDCT,DDCT)
EDCT=INTGRL(EDCT,EDCT)
FDDCT=INTGRL(FDDCT,FDDCT)
A=INTGRL(0.,ADCT)
B=INTGRL(0.,BDCT)
C=INTGRL(0.,CDCT)
D=INTGRL(0.,DDCT)
EE=INTGRL(0.,EDCT)
FF=INTGRL(0.,FDCT)
*SECTION 5 - OUTPUT FCHARACTERISTICS
  TIMER1=0.01,FINTIM=240.0,OUTDEL=1.C,PRDEL=1.C
  END
  PARAN DR=-0.4365
  END
  PARAN DR=-0.6111
  END
  STCF

```



```

AAA=XUDCT-NL
AAB=0.
AAC=0.
AAD=0.
AEE=0.
AAF=0.
AEA=0.
AEC=0.0
AED=0.0
AEE=NVDOT-ML
AEB=YVDOT-ML
ACB=0.
ACC=1.0
ACD=0.
ACE=0.
ACF=0.
ACA=0.0
ACB=0.0
ACC=0.
ACD=1.0
ACE=0.0
ADF=0.0
AEA=0.
AEB=0.
AEC=0.
AED=0.
AEE=1.0
AEF=0.
AFA=0.
AFB=LC*YRDOT
AFC=0.0
AFE=0.
AFF=NRDOT-IZ
AFF SECTION 2A
DEL=VALUE(AAA,0,0)
CCFAA=VALUE(AAA,1,1)
CCFAB=VALUE(AAA,2,1)
CCFAC=VALUE(AAA,3,1)
CCFAD=VALUE(AAA,4,1)
CCFAE=VALUE(AAA,5,1)
CCFAF=VALUE(AAA,6,1)
CCFBA=VALUE(AAA,1,2)
CCFB0=VALUE(AAA,2,2)
CCFBC=VALUE(AAA,3,2)
CCFBD=VALUE(AAA,4,2)

```



```

CC FBE=VALUE (AAA, 5, 2)
CC FBF=VALUE (AAA, 6, 2)
CC FCA=VALUE (AAA, 1, 3)
CC FCCB=VALUE (AAA, 2, 3)
CC FCCC=VALUE (AAA, 3, 3)
CC FCCD=VALUE (AAA, 4, 3)
CC FCCE=VALUE (AAA, 5, 3)
CC FFDA=VALUE (AAA, 6, 3)
CC FFB=VALUE (AAA, 1, 4)
CC FDC=VALUE (AAA, 2, 4)
CC FDD=VALUE (AAA, 3, 4)
CC FDE=VALUE (AAA, 4, 4)
CC FDF=VALUE (AAA, 5, 4)
CC FEA=VALUE (AAA, 6, 4)
CC FEB=VALUE (AAA, 1, 5)
CC FEC=VALUE (AAA, 2, 5)
CC FED=VALUE (AAA, 3, 5)
CC FEF=VALUE (AAA, 4, 5)
CC FEF=VALUE (AAA, 5, 5)
CC FFFA=VALUE (AAA, 6, 5)
CC FFB=VALUE (AAA, 1, 6)
CC FFC=VALUE (AAA, 2, 6)
CC FFD=VALUE (AAA, 3, 6)
CC FFE=VALUE (AAA, 4, 6)
CC FFF=VALUE (AAA, 5, 6)
CC FFF=VALUE (AAA, 6, 6)

```

DYNAMIC

CC=DR\*67 • 273

```

X=TIME
KA1=-X*DR*U*U*DR/LC
KB1=-Y*DR*U*U/LC
KC1=-K*DR*U*U/LC**2
KF1=-ND*F*U*U/LC**2
UE=U*YV/LC
UE=U*KV/LC**2
BF=U*NV/LC**2
BE=U*YP
EC=U*KP/LC
BC=U*NP/LC
BF=U*YR
BC=U*KR/LC
BF=U*NR/LC

```

\*SECTICN 3-DEFINITIONS  
L=ADOT  
V=BDOT

```

L=ADOT
V=BDOT
V=BDOT
V=BDOT
V=BDOT

```







```

DO 33 J=LP1,N
33 X(K,J)=X(K,J)-RATIO*X(L,J)
CCNT NUE
34 CC 41 K=1,N
41 CC=DD*X(K,K)
D=DD
VALUE=D
WRITE(6,52),I,M,VALUE
52 FCNRMAT(,C0F,I1,I1,'=I,EL5.6)
      RETURN
      END JCB

```



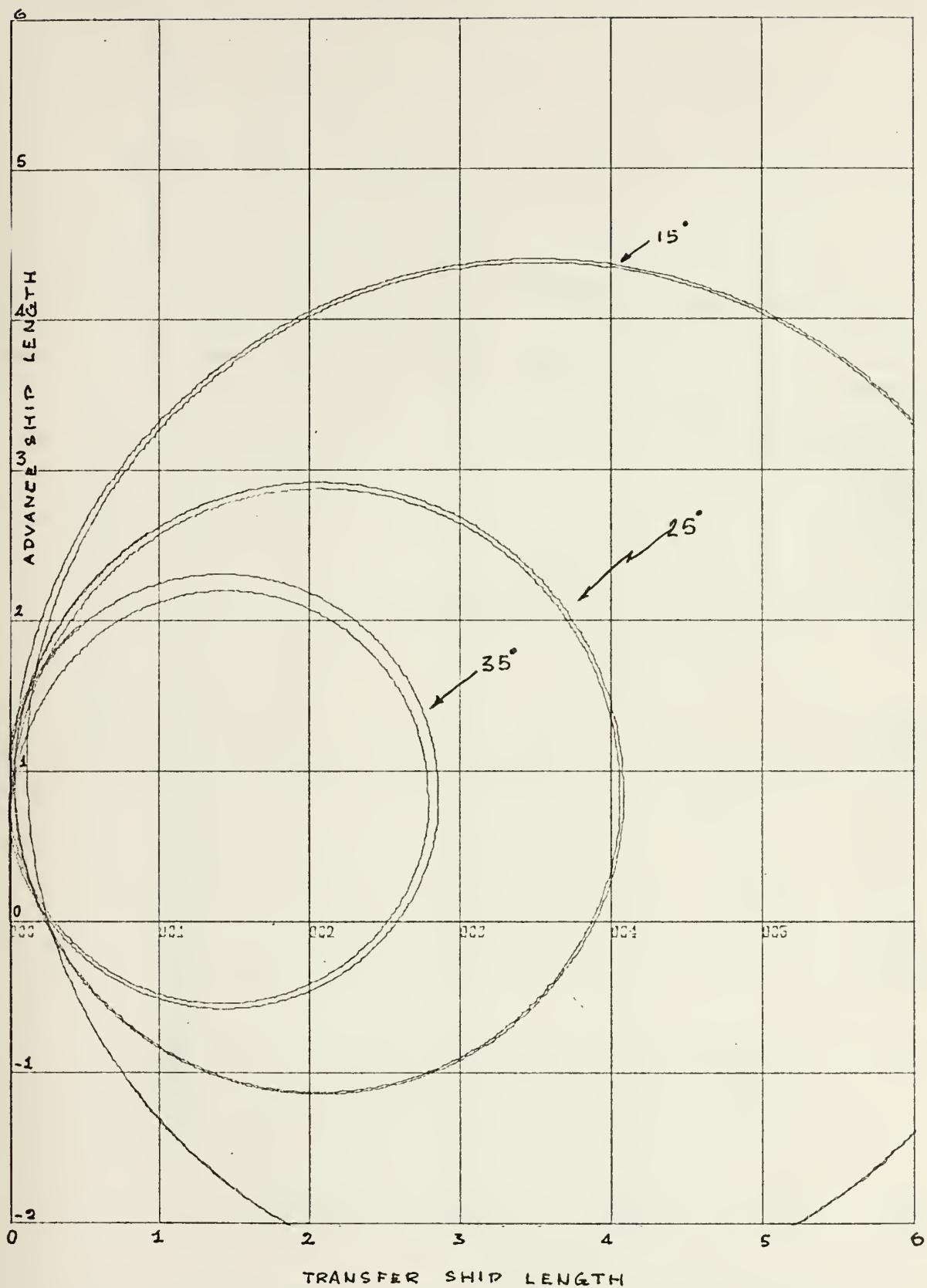


FIG. 8 ADVANCE VS. TRANSFER SHIP LENGTH (DR = 15°, 25° & 35°)



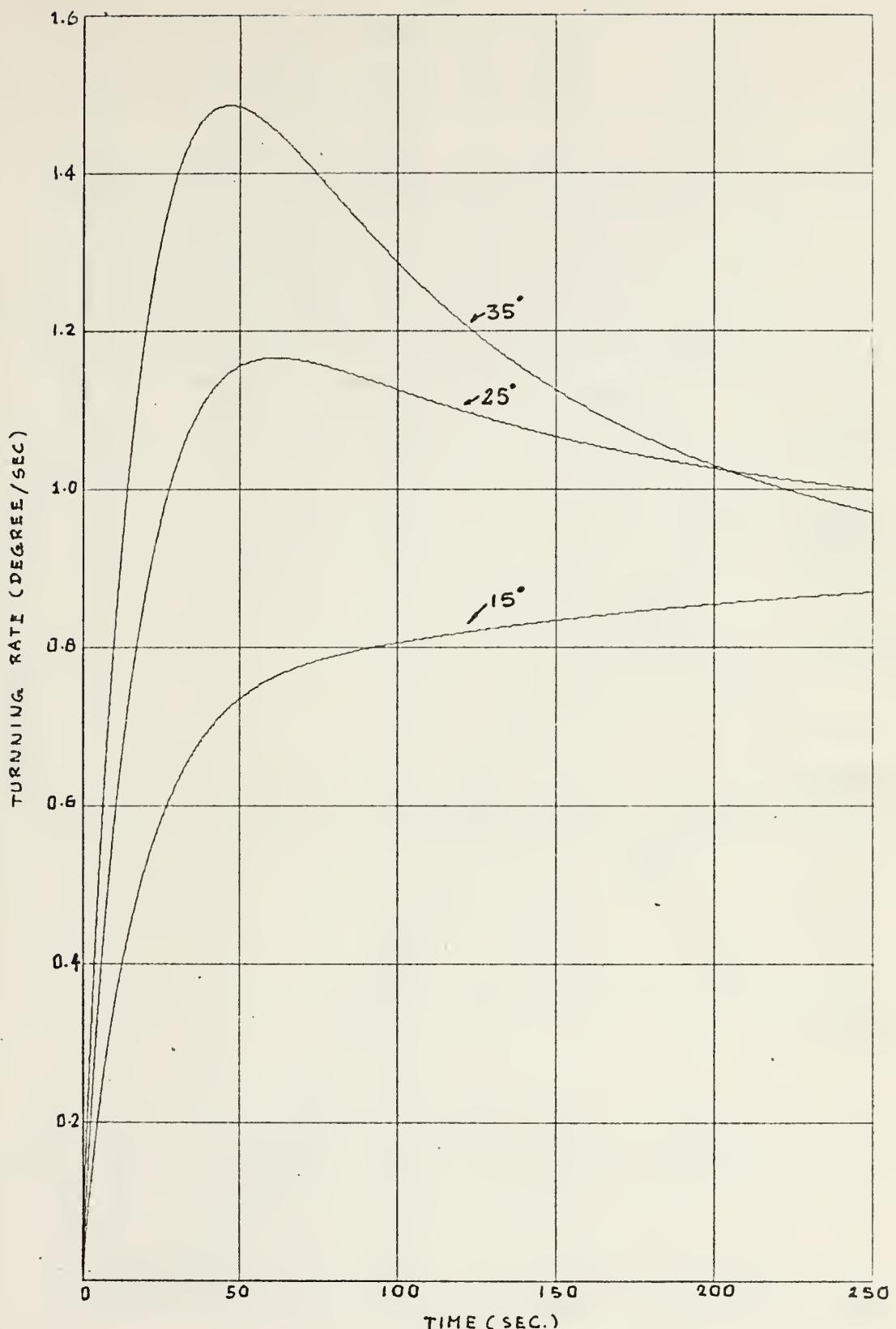


FIG. 9 TURNNING RATE AS A FUNCTION OF TIME  
(RUDDER 15°, 25 & 35°)



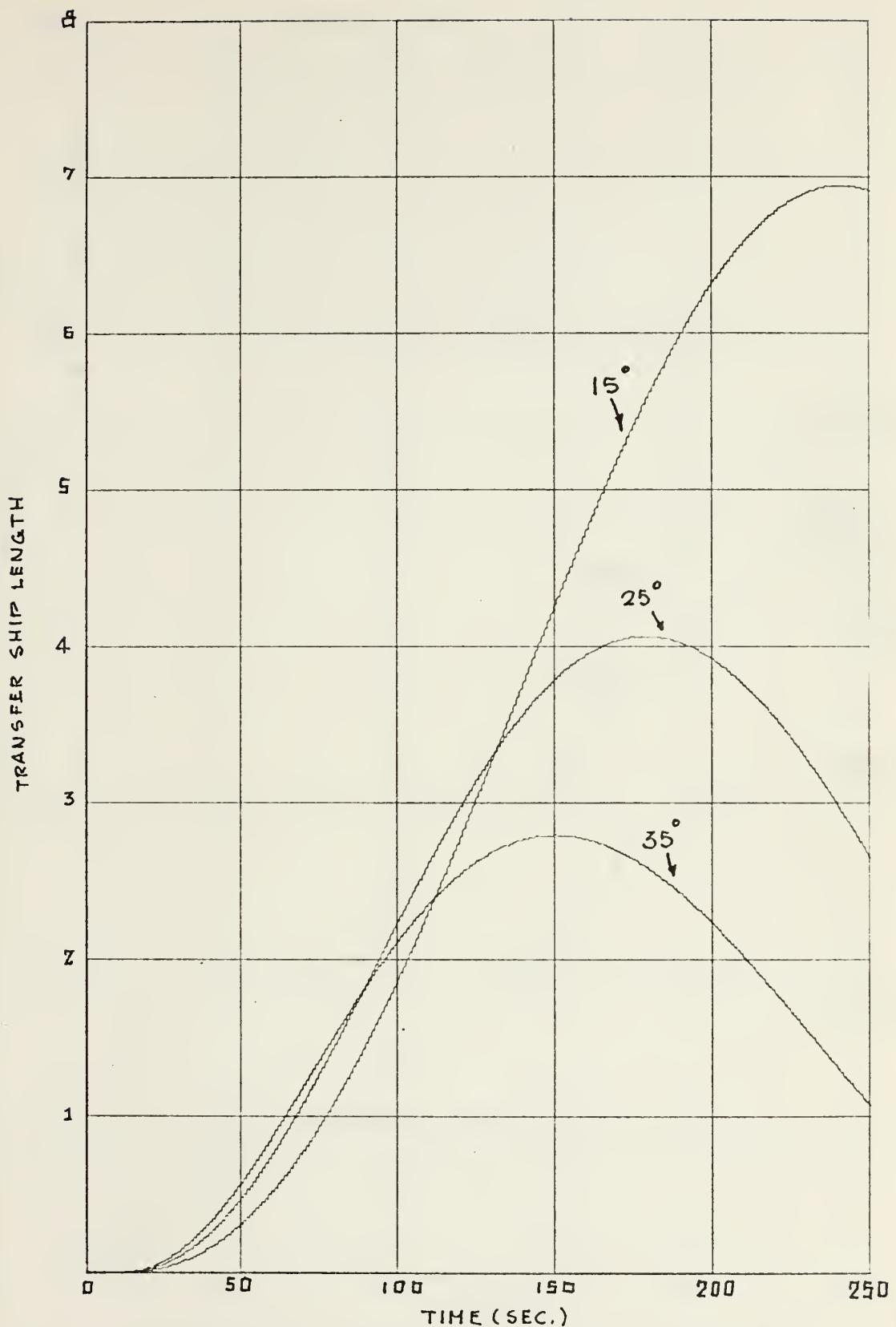


FIG. 10 TRANSFER SHIP LENGTH VS. TIME  
(RUDDER 15°, 25° & 35°)



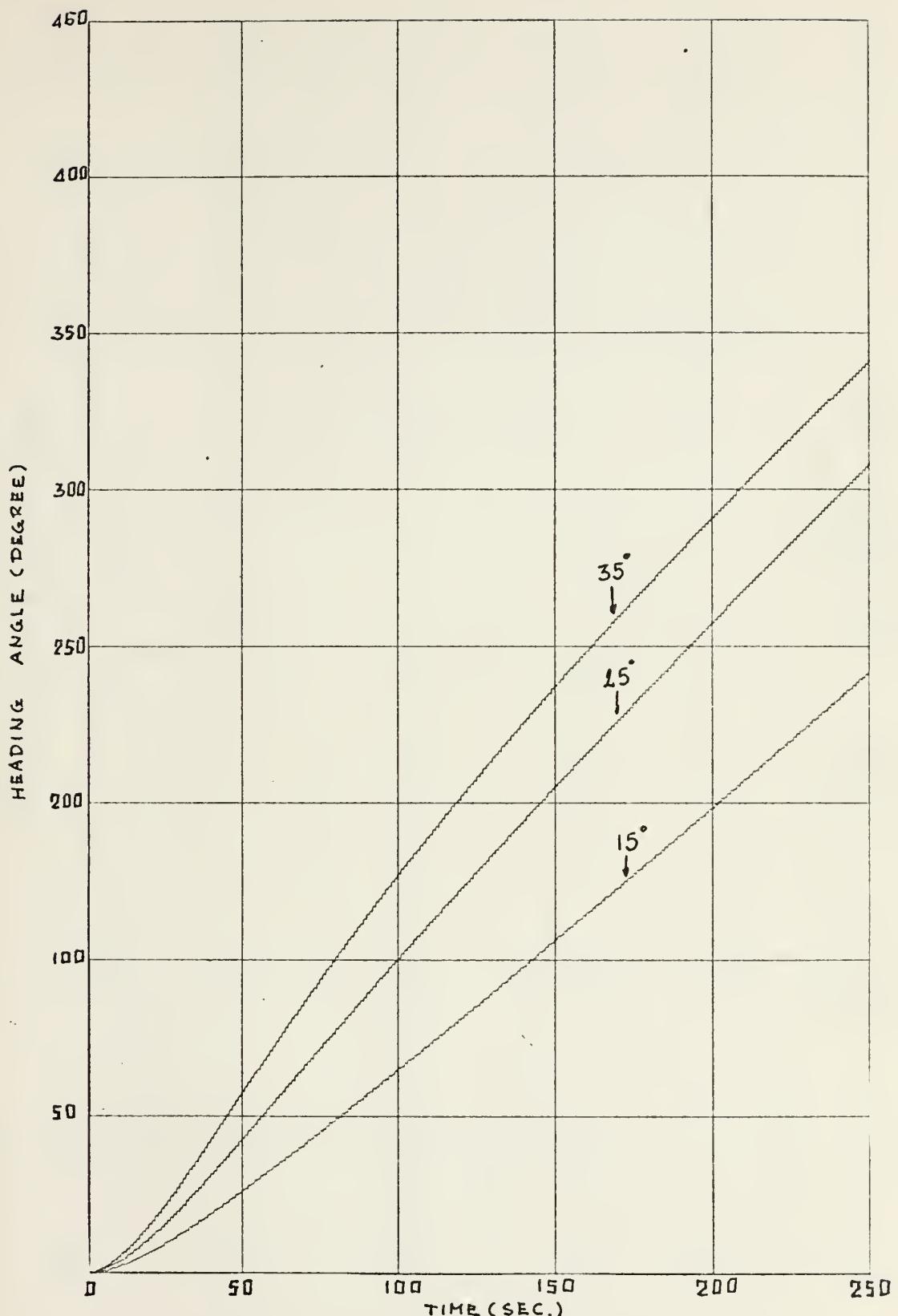


FIG. II HEADING ANGLE AS A FUNCTION OF TIME  
(RUDDER 15°, 25° & 35°)



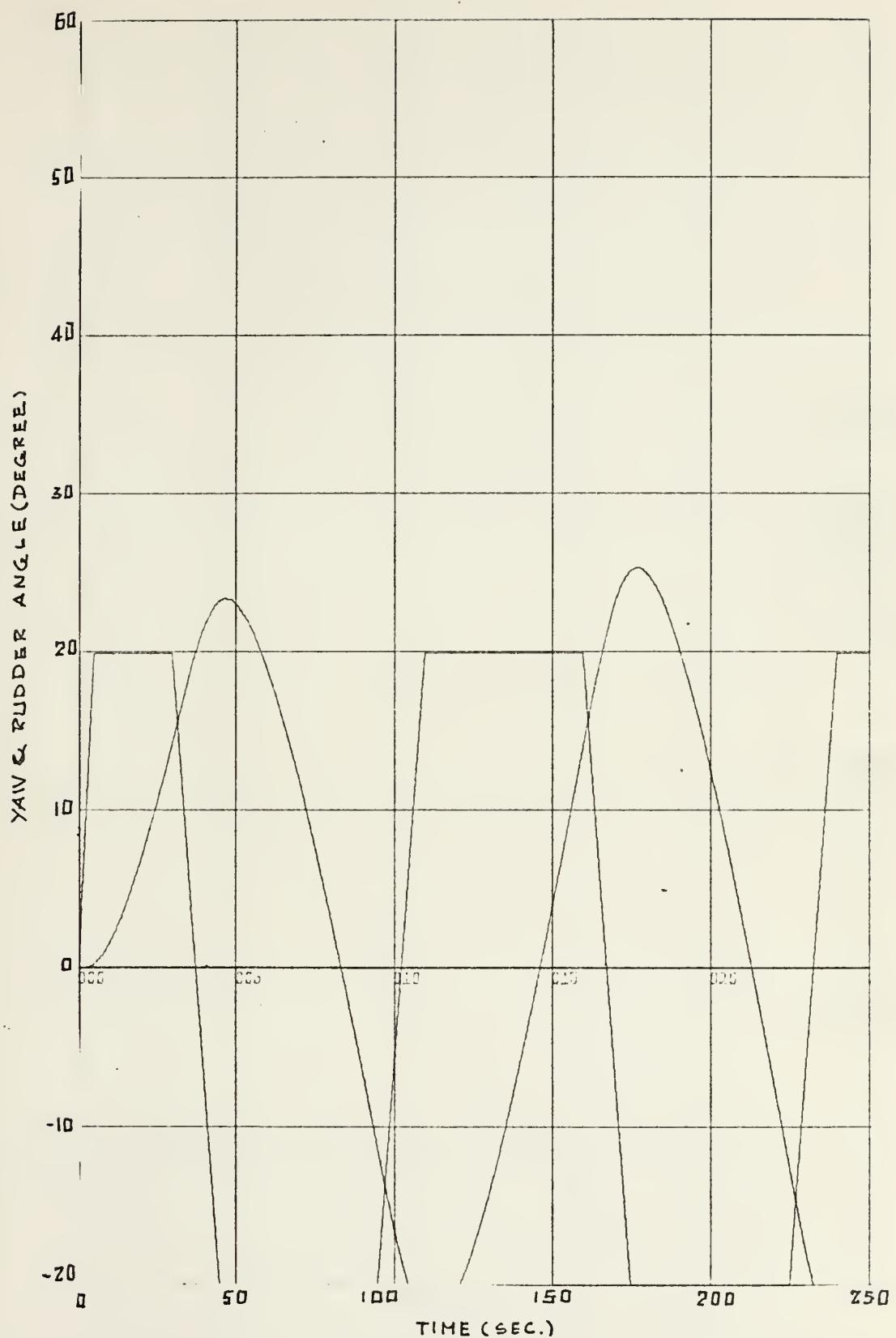


FIG. 12 YAW & RUDDER ANGLE VS. TIME  
(ZIG-ZAG MANOEUVRE)



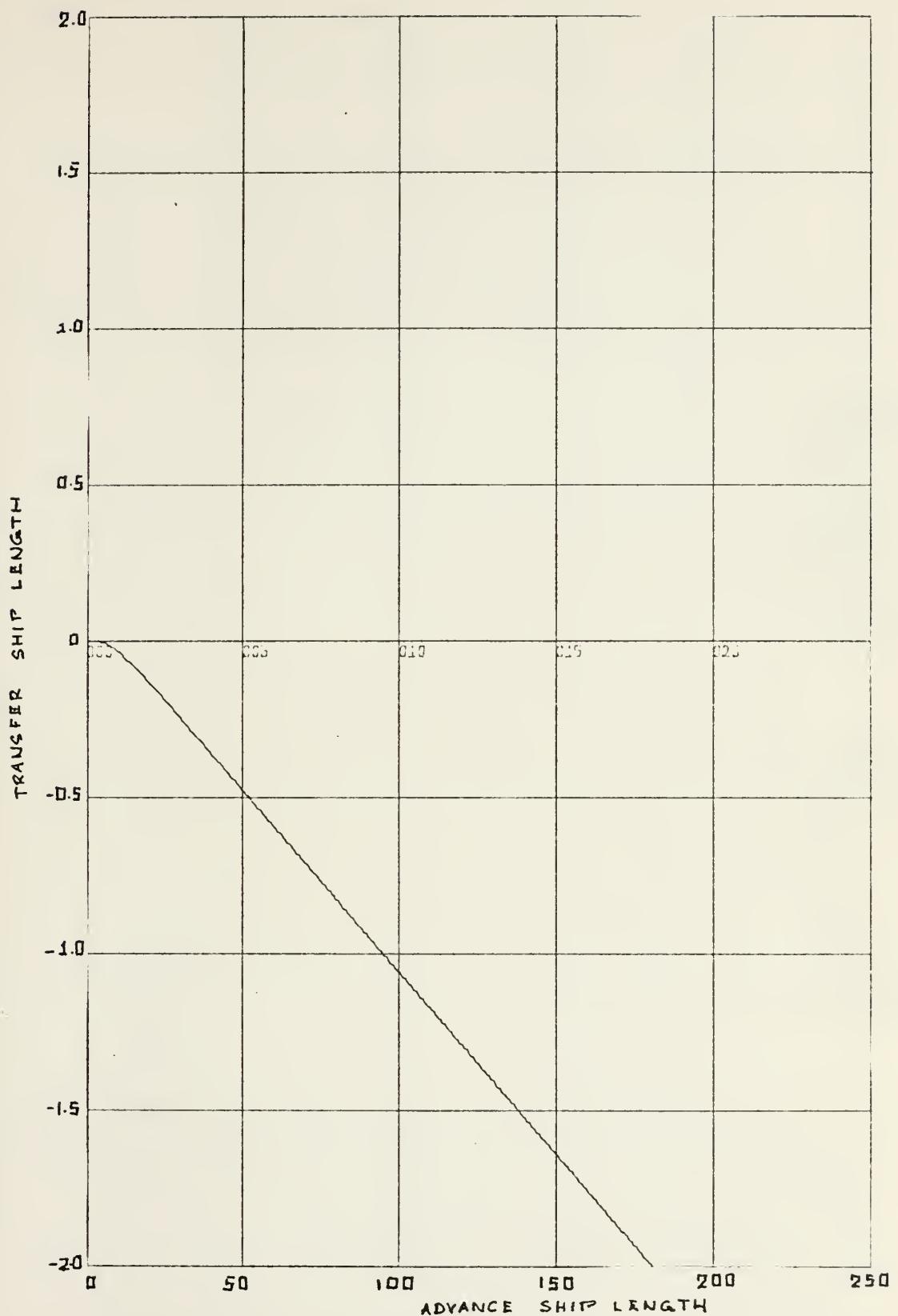


FIG. 13 DIRECTION OF THE SHIP WHEN EXTERNAL MOMENT FORCE APPLIED TO THE SHIP



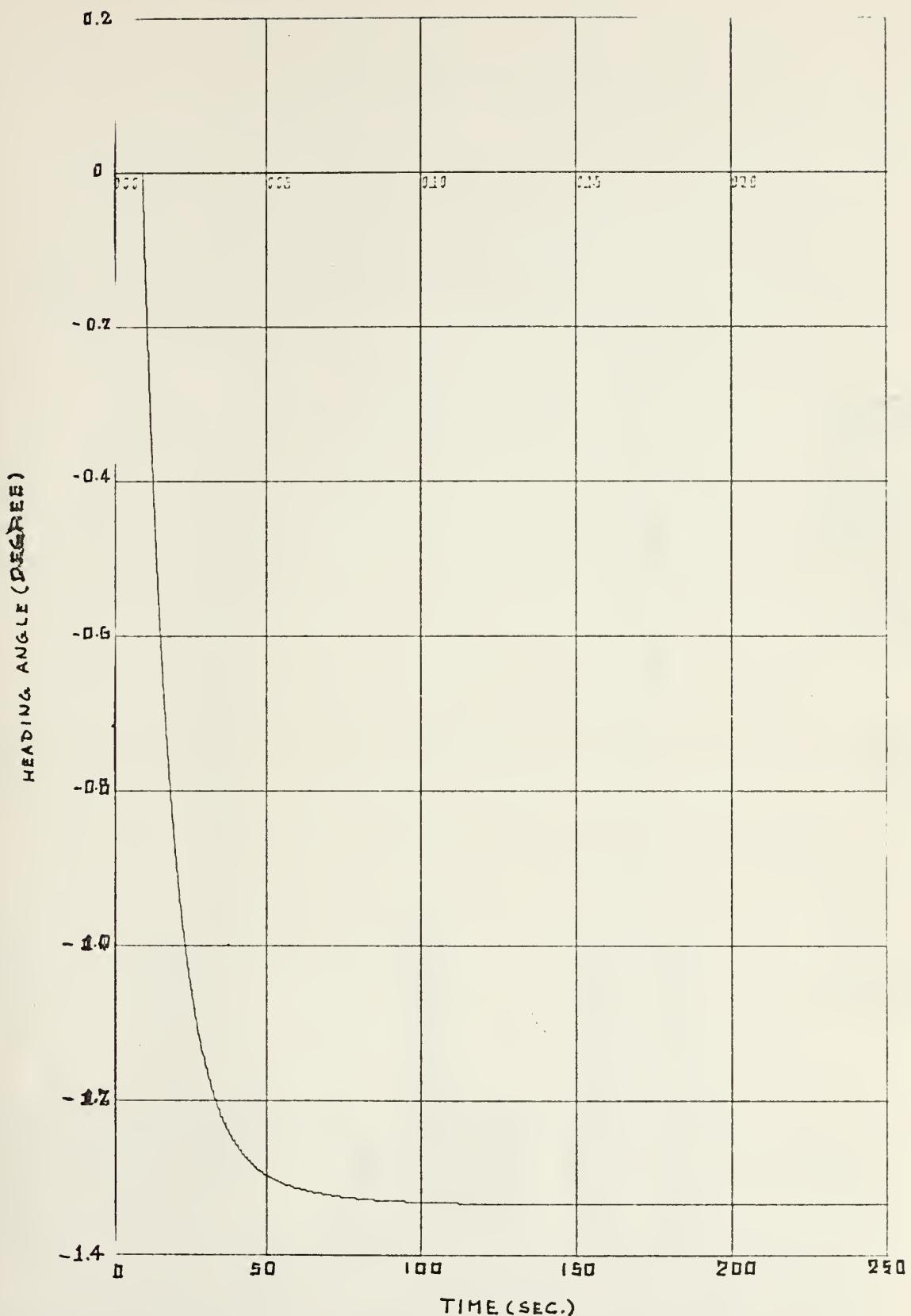


FIG. 14 HEADING ANGLE VS. TIME  
(APPLIED FORCING MOMENT TO THE SHIP)







17  $\frac{\text{IF}(\text{NCPLCT})}{\text{STCP}}$  17,17,16  
EAC



#### IV. CONCLUSIONS

The equation of motion of surface ship and computer program developed here including all of six degrees of freedom, but the study in III concerns only three degrees of freedom (surge, sway, yaw) because hydrodynamic coefficients are not available; when the state of the art reaches the stage in which hydrodynamics coefficients are available, this computer program can be used in all six degrees of freedom.

Some results from III are not too perfect because the lack of some constants and coefficients such as the value of mass ( $m$ ), initial velocity (ADOTO), command speed (UC), etc. But for study can adjust from curve for the model test [Ref. 6].

This computer program did not include some external effects such as effects of wave and wind, but these effects could be included in the program by adding terms to the IF equation.

The following implementations are suggested for the future work.

- A. Study all six degrees of freedom.
- B. Study for the effects of waves and wind.
- C. Study for control of the velocity and direction of the ship (by use of "MACROS", "PROCEDURE" or subprogram in CSMP).



## LIST OF REFERENCES

1. Abkowitz, Martin A.: Stability and Motion Control of Ocean Vehicles. The MIT Press, 1969.
2. George J. Thaler: Ship Control System. Naval Post-graduate School, 1973.
3. System/360 Continuous System Modeling Program, the IBM Press GH20-0367-4.
4. Edgar Romero: Mathematical Models and Computer Solution for the Equations of Motion of Surface Ships and Submarines, in Six Degrees of Freedom. Thesis, Naval Postgraduate School, 1972.
5. Technical and Research Bulletin No. 1-5, Society of Naval Architect and Marine Engineers.
6. Report for ship "D" of NSRDC (Naval Ship Research and Development Center).



INITIAL DISTRIBUTION LIST

No. Copies

1. Defense Documentation Center Cameron Station Alexandria, Virginia 22314	2
2. Library, Code 0212 Naval Postgraduate School Monterey, California 93940	2
3. Professor M. L. Wilcox, Code 52Wx Department of Electrical Engineering Naval Postgraduate School Monterey, California 93940	2
4. Dr. G. J. Thaler, Code 52Tr Department of Electrical Engineering Naval Postgraduate School Monterey, California 93940	1
5. Professor A. Gerba, Jr., Code 52Gz Department of Electrical Engineering Naval Postgraduate School Monterey, California 93940	1
6. Lieutenant Commander Aporn Ratanaruang Royal Thai Navy Bangkok, Thailand	3
7. Professor D. M. Layton, Code 57Ln Aeronautical Engineering Department Naval Postgraduate School Monterey, California 93940	1
8. Library, Planning & Design Division Thai Naval Shipyard Bangkok, Thailand	1
9. Education Section Personal Department Royal Thai Navy Bangkok, Thailand	1



## DOCUMENT CONTROL DATA - R &amp; D

(Security classification of title, body of abstract and indexing annotation must be entered when the overall report is classified)

ORIGINATING ACTIVITY (Corporate author)  
Naval Postgraduate School  
Monterey, California 939402a. REPORT SECURITY CLASSIFICATION  
Unclassified  
2b. GROUP

REPORT TITLE

Digital Computer Simulation for Surface Ship Control

DESCRIPTIVE NOTES (Type of report and, inclusive dates)

Master's Thesis; June 1973

AUTHOR(S) (First name, middle initial, last name)

Aporn Ratanaruang

REPORT DATE June 1973	7a. TOTAL NO. OF PAGES 67	7b. NO. OF REFS 6
CONTRACT OR GRANT NO.	8a. ORIGINATOR'S REPORT NUMBER(S)	
PROJECT NO.	9b. OTHER REPORT NO(S) (Any other numbers that may be assigned this report)	

DISTRIBUTION STATEMENT

Approved for public release; distribution unlimited.

SUPPLEMENTARY NOTES	12. SPONSORING MILITARY ACTIVITY Naval Postgraduate School Monterey, California 93940
---------------------	---

ABSTRACT

The general equations of surface ship motion are developed and standardized for simulation in digital computer. Digital simulations of the dynamics of the surface ship in three degrees of freedom are done with and without non-linear and cross-coupling terms.



KEY WORDS	LINK A		LINK B		LINK C	
	ROLE	WT	ROLE	WT	ROLE	WT
ship control						
surface ship in six degrees of freedom						
ship maneuvering						
simulation						



Thesis 145211  
R24 Ratanaruang  
c.1 Digital computer simu-  
lation for surface ship  
control.  
14 NOV 73 20722  
3 Feb 68 26150  
2 Feb 68 27780  
3 1350

Thesis 145211  
R24 Ratanaruang  
c.1 Digital computer simu-  
lation for surface ship  
control.

thesR24  
Digital computer simulation for surface



3 2768 001 01336 0  
DUDLEY KNOX LIBRARY